

STRESS ANALYSIS OF CERAMIC TURBINE
BLADES BY FINITE ELEMENT
METHOD - PART I.

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NAVAL POSTGRADUATE SCHOOL

Monterey, California



THESIS

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BLADES BY FINITE ELEMENT METHOD - PART I

by

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Stress Analysis of Ceramic Turbine Blades
by Finite Element Method - Part I

by

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Lieutenant Commander,¹¹ United States Navy
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ABSTRACT

The search for more efficient gas turbine engines has led to the proposal for the replacement of metal high temperature components with ceramic components. Essential to this effort is the numerical analysis of proposed designs. This thesis report describes the model discretization of a proposed blade design, the development of pre- and post-processors for the ADINA finite element code and the initial stress analysis of the modeled blade.

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I. INTRODUCTION

The need to increase fuel efficiency of gas turbines has led to a sizable research investment into suitable materials and designs of ceramic gas turbine components to replace high temperature resistant superalloys. This replacement is envisioned to yield a threefold benefit:

1) The refractory characteristics of ceramics will allow a greater source-sink temperature differential increasing plant efficiency without penalizing overall performance by requiring increased cooling air bled off the compressor.

2) The lower specific weight of ceramics relative to the metal components to be replaced will lower plant weight thereby increasing the plant's power-to-weight ratio.

3) Ceramic constituent components are, in general, domestically available giving them a strategic edge over superalloys.

The basic problem preventing realization of these benefits is the brittle characteristic of ceramics. Since the brittle property is associated with the desirable refractory characteristic, material research and development can only be expected to yield marginally better ceramics in the near future. Any short term success will, therefore, be a result of design innovations which will compensate for the brittle nature of ceramics to allow their use for components which have formerly been engineered on the basis of use of ductile materials.

Essential to this design effort is the use of numerical techniques to model proposed designs and develop alternative criteria with the ultimate goal of minimizing stress concentrations which key catastrophic brittle failure. This thesis describes the efforts of the author to utilize the computer hardware and software available at the Naval Postgraduate School (NPS) to analyze stress distributions in gas turbine ceramic replacement components. Time constraints dictated limiting the scope of analysis to the modeling and development of pre- and post-processors for a proposed first stage ceramic turbine blade subjected to centrifugal loading.

A turbine blade design, developed by the AIRESEARCH division of GARRETT CORPORATION under a Navy managed contract for development of a one-hundred hour test gas turbine engine using ceramic components for the combustor, stator vanes, and rotor blades, was used in this work. Stress distribution analyses were centered around the "Automatic Dynamic Incremental Nonlinear Analysis" (ADINA) code developed by Dr. Klaus-Jurgen Bathe of the Massachusetts Institute of Technology [Ref. 1] as implemented on the NPS IBM 360-67 computer system. The bulk of the modeling effort was accomplished on a Hewlett-Packard 9830 desk top calculator with associated graphics equipment and the PSAP1 graphics package [Ref. 2] implemented on the NPS computer by Lt. Adrian Kibler. Material properties were those of hot-pressed Silicon-Nitride provided by AIRESEARCH. Post-processing graphics utilized a

contour plotting routine developed by Gary L. Giles of Langley Research Center [Ref. 3] and implemented on the NPS computer by Dr. Gilles Cantin of NPS and the author.

The complete analytical efforts were divided into four major categories:

- 1) development of a discretized model of the blade and attachment design.
- 2) development of a pre-processor to calculate consistent centrifugal loads.
- 3) modification of the ADINA code to yield a maximum of stress output locations.
- 4) execution of ADINA with model developed and elaboration of post-processors to help in the interpretation of results.

II. ANALYTICAL MODEL DEVELOPMENT

Developing the finite element mesh of the complex geometric shapes incorporated in the blade was a compromise between geometric accuracy, mathematical compatibility and economy of effort. Using drawings provided by AIRESEARCH [Refs. 4, 5, 6], a mathematical definition of the airfoil and the root was developed separately and then mated using twenty node bricks arranged compatibly throughout the entire assembled mesh.

A. BLADE NOMENCLATURE

Figure 1 illustrates the nomenclature used in this paper to describe the gas turbine blade model and the reference system used to define the geometry. The portion of the component aerodynamically designed to be in the gas flow is termed the airfoil. The fillet is the transition section in between the airfoil and the attachment root designed to minimize stress concentrations. The attachment root (also termed the dovetail) forms the base of the airfoil, provides radial and tangential placement of the blade, and transmits the forces developed by the gas flow to the disk. A right-hand cartesian coordinate system was used to describe the geometry. The X-axis coincided with the turbine axis-of-rotation. The Z-axis coincided with the stacking axis for the airfoil profiles. The Y-axis was then mutually perpendicular to the X- and Z-axis, with the system origin on the rotation axis.

B. GEOMETRIC DEFINITION

1. Airfoil

Reference 4 provided sixty-two perimeter points for x-y plane cross-sections at ten z-levels from $z=3.1$ to $z=4.0$ inches. Linear fairing was prescribed for machining between cross-sections in order to facilitate manufacture. The airfoil tip level of 3.874 inches and z-level 3.385 inches (level of beginning of fillet definition) were plotted and stored for use in discretizing the airfoil. Because of the linear fairing technique, intermediary points were left to the ADINA code feature of node generation for definition.

2. Fillet

The fillet geometry consisted of two radii; 0.3 inch and 0.1 inch. The locus of centers of the 0.3 inch part of the fillet was at a constant level of 3.385 inches. The arc subscribed was tangent to the linear airfoil. The locus of centers of the 0.1 inch radius arc was centered such that the arc subscribed was tangent to both the 0.3 inch radius fillet section and the base which was considered to be a 3.2 inch radius right circular cylinder about the axis of rotation. This double radius design was chosen in order to reduce the possibility of stress concentrations by approximating an elliptic geometry.

Geometric definition of the fillet was made in sixty-two, two-dimensional planes parallel to the Z-axis and containing the normal vector to each of the sixty-two

defined perimeter points at $z=3.385$ inches (Figure 2a). The following described algorithm was developed to define the limiting points illustrated by Figure 2b.

a. Determination of the External Normal

A second degree Lagrange interpolating polynomial was passed through three successive perimeter points at the cross-section $z=3.385$ inches, and the first derivative was evaluated for the middle point in order to determine the slope of the tangent vector at this point. The negative reciprocal of the derivative is the slope of the line normal to the analysis point. Taking the y-component as -1 and the x-component as the derivative (dy/dx), a normal vector was defined. This vector was then normalized to give a unit vector. The analysis point was tested to determine the desired coefficient signs in order to define the normal with the direction external to the body of the airfoil. This procedure was carried out for each perimeter point, thereby defining sixty-two normals from the airfoil.

b. Locus of 0.3 Inch Radius Centers

The coordinate system was translated and rotated to yield a working plane which was parallel to the Z-axis and contained the unit normal vector for the point being analyzed. The curvature center was adjusted in this working plane at $z=3.385$ inches such that the 0.3 inch arc was tangent to the linear airfoil surface. These contact points, defining the start of fillet curvature, were transformed to the original coordinate system for future use.

c. Locus of 0.1 Inch Radius Centers

The 0.1 inch radius curvature centers were defined in each of the sixty-two working planes by use of a Newton iteration scheme to adjust the center of curvature such that the subscribed arc was tangent to both the 0.3 inch arc and the top of the root. These points of tangencies were determined, transformed to the original coordinate system and stored.

d. Definition of Perimeter Point at Arbitrary Z-Level

Using the above determined points (Figure 2b), an algorithm was developed which took as input any desired z-level in the fillet region, tested for the correct definition of geometry for that level and yielded an array of sixty-two perimeter surface points for that level.

Information from the above procedure allowed the definition of a sixty-two point perimeter at any z-level of an airfoil or fillet cross-section parallel to the x-y plane.

3. Attachment Root

The attachment root geometry was defined in a vertical plane containing the stacking axis and perpendicular to the horizontal longitudinal centerline of the body of the root which has a broach angle of 23° relative to the axis of rotation (Figure 3). Nine different geometries consisting of straight lines and arcs of circles were defined by Ref. 6 in this plane for the perimeter of the root. These geometries were mathematically defined, and the limiting point coordinates of each segment were determined. This

shape was projected into the fore and aft faces of the disk. Symmetry conditions allowed simple manipulations of coordinates in order to define the shape of the entire attachment root. An algorithm was developed to define 231 points on any desired horizontal surface.

C. ANALYSIS MESH DISCRETIZATION

Major considerations in the development of the analysis model were to define a mesh which adequately described the geometry of the blade and yielded sufficient data points for meaningful evaluation, without making the mesh so fine as to require an inordinate amount of computer time for solution. Twenty node bricks were chosen for use throughout the mesh because of their capability of accurately defining any second order geometric curve (used almost exclusively by the designers of the blade) and their isotropic sensitivity to the applied loads.

The airfoil was represented by a mesh of twelve elements arranged one deep, three high and four long (Figure 4). The top two layers defined the linearly faired section, and the bottom row defined the fillet region. Choice of nodal point coordinates were made with the assistance of plots of horizontal cross-sections of points defined by geometry definition algorithms.

Chosen for the attachment root representation were nominal three deep, four long layers of elements defining each different vertical geometric segment (Figure 5). The exact

make-up of the root mesh was controlled by the mating of the airfoil with the attachment root which caused significant distortions because the design definition of the fillet base extended beyond the limits of the top surface of the attachment root. This mis-match of geometries necessitated brute force adjustment of nodal point coordinates in order to mate the two regions into a single consistent mesh.

Once the mating process was accomplished, all the chosen nodal points and the element connectivities were card punched in the format required by ADINA. The ADINA deck was then used as input to the PSAP1 graphics package in order to de-bug the mesh. After adjustments were made to yield a visually satisfactory mesh, a trial analysis was run which brought to light numerous mathematical difficulties caused by the distortions resulting from the mating of dissimilar geometries. These were corrected, and the final mesh (Figure 6) was considered ready for analysis.

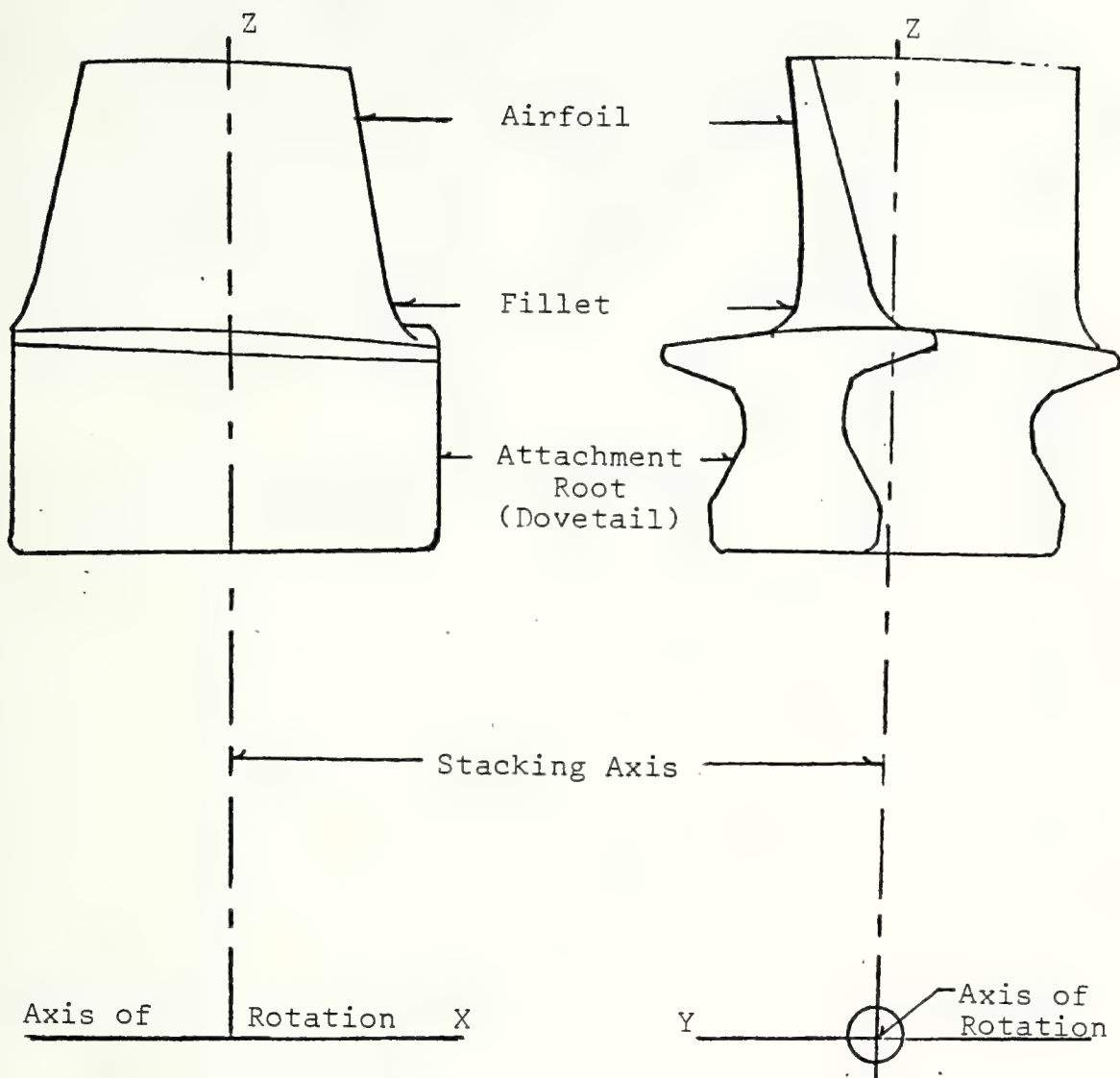
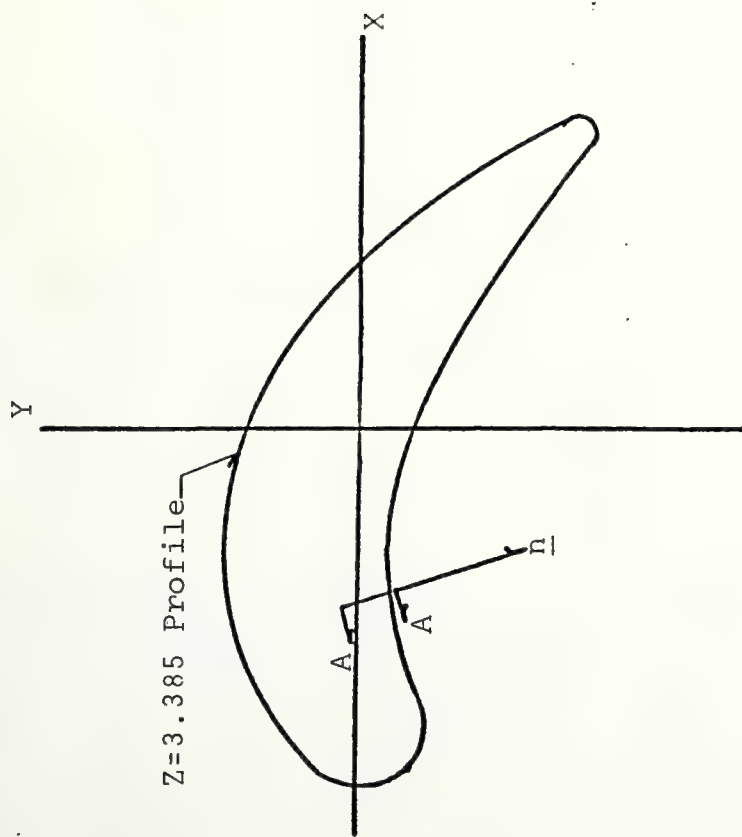
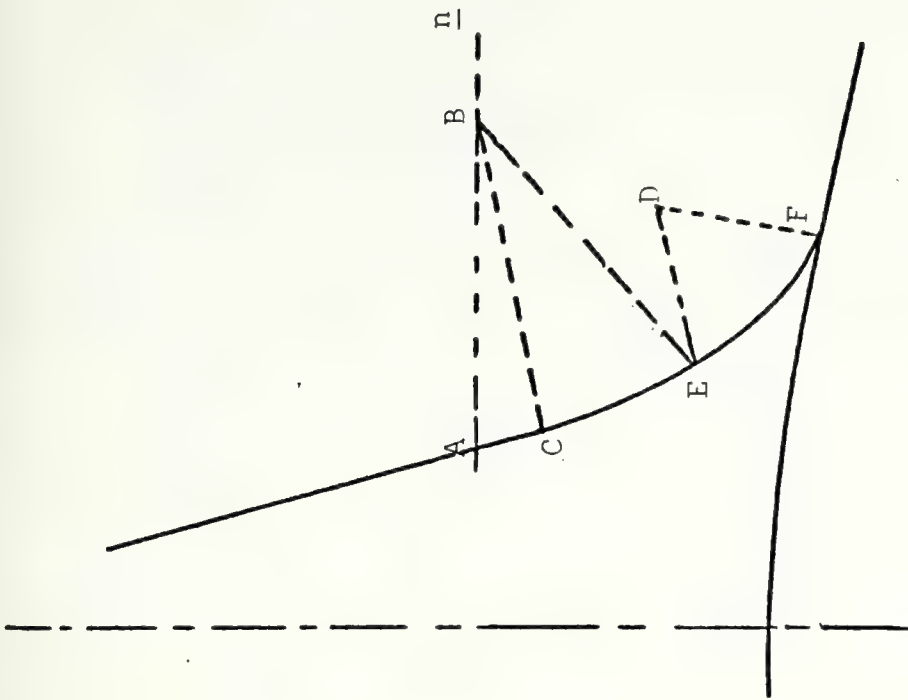


Figure 1. Illustration of Nomenclature and Reference System of Turbine Rotor Blade.



(a)



Section A-A

(b)

Figure 2. Illustration of Plane in Which Fillet Geometry is Defined.
Page 1 of 2

Definition of nomenclature for Figure 2

n = outward pointing normal

A = level $z = 3.385$ inches

B = center of 0.3 inch fillet radius curvature

C = point of tangency between linear airfoil and fillet
curvature

D = center of 0.1 inch fillet radius curvature

E = point of tangency between 0.3 inch fillet curvature
and 0.1 inch fillet curvature

F = point of tangency between 0.1 inch fillet curvature
and attachment root surface

Figure 2. Continued, Page 2 of 2.

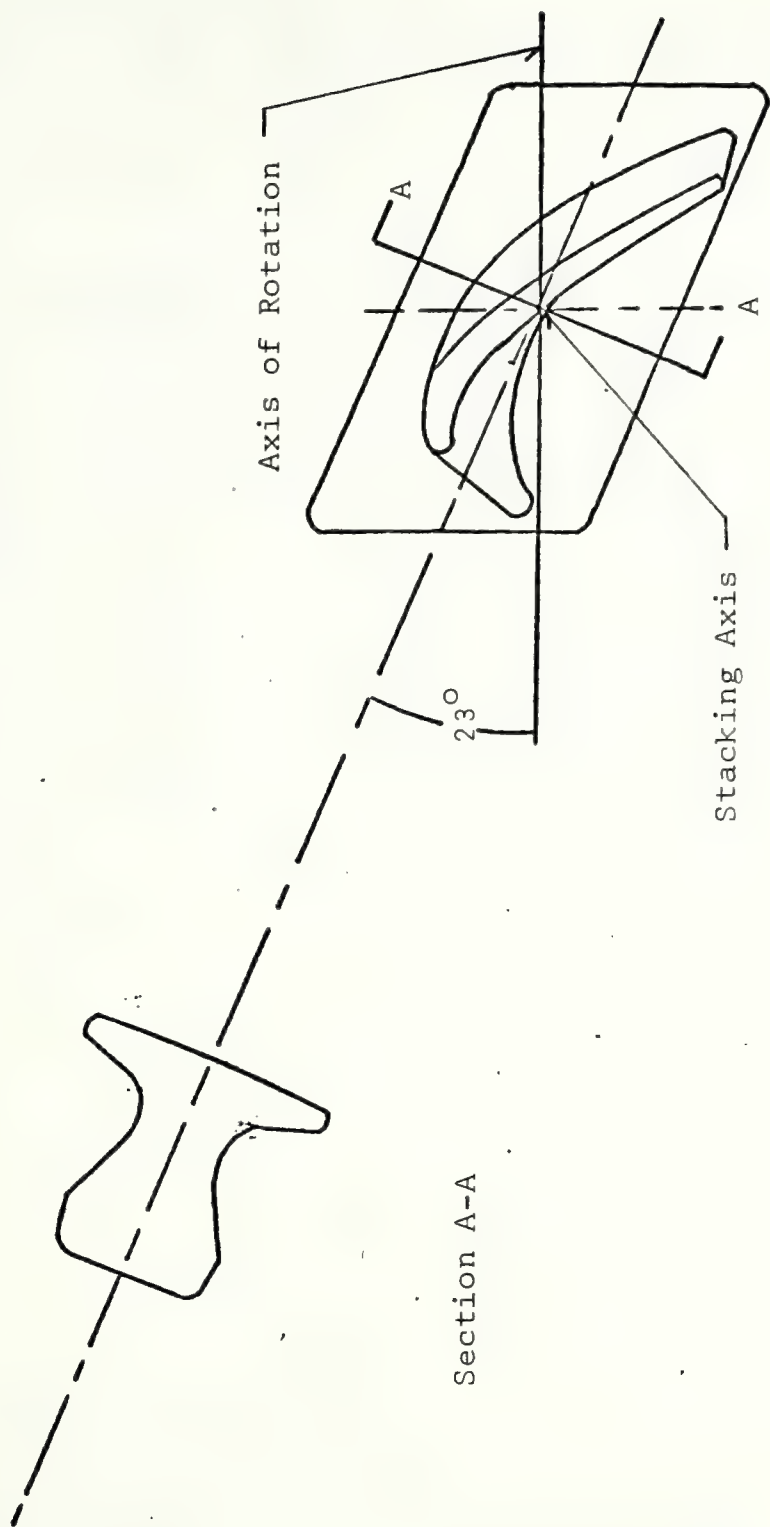


Figure 3. Illustration of Plane in Which Attachment Root Geometry is Defined.

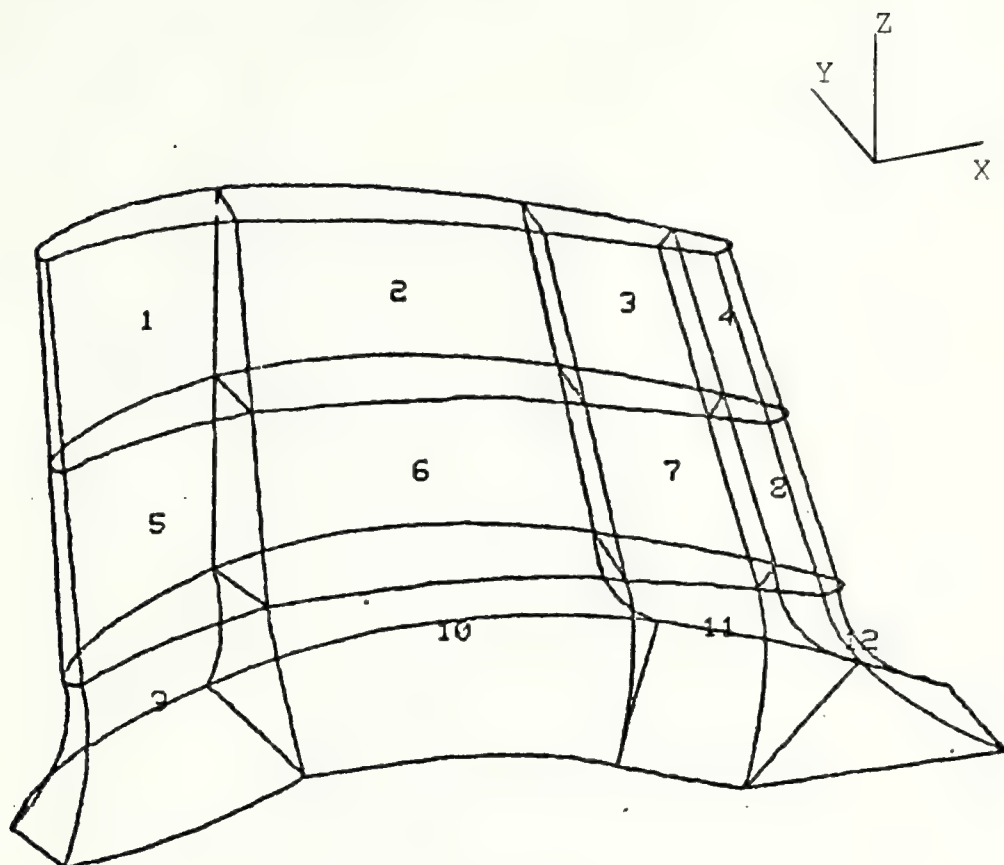
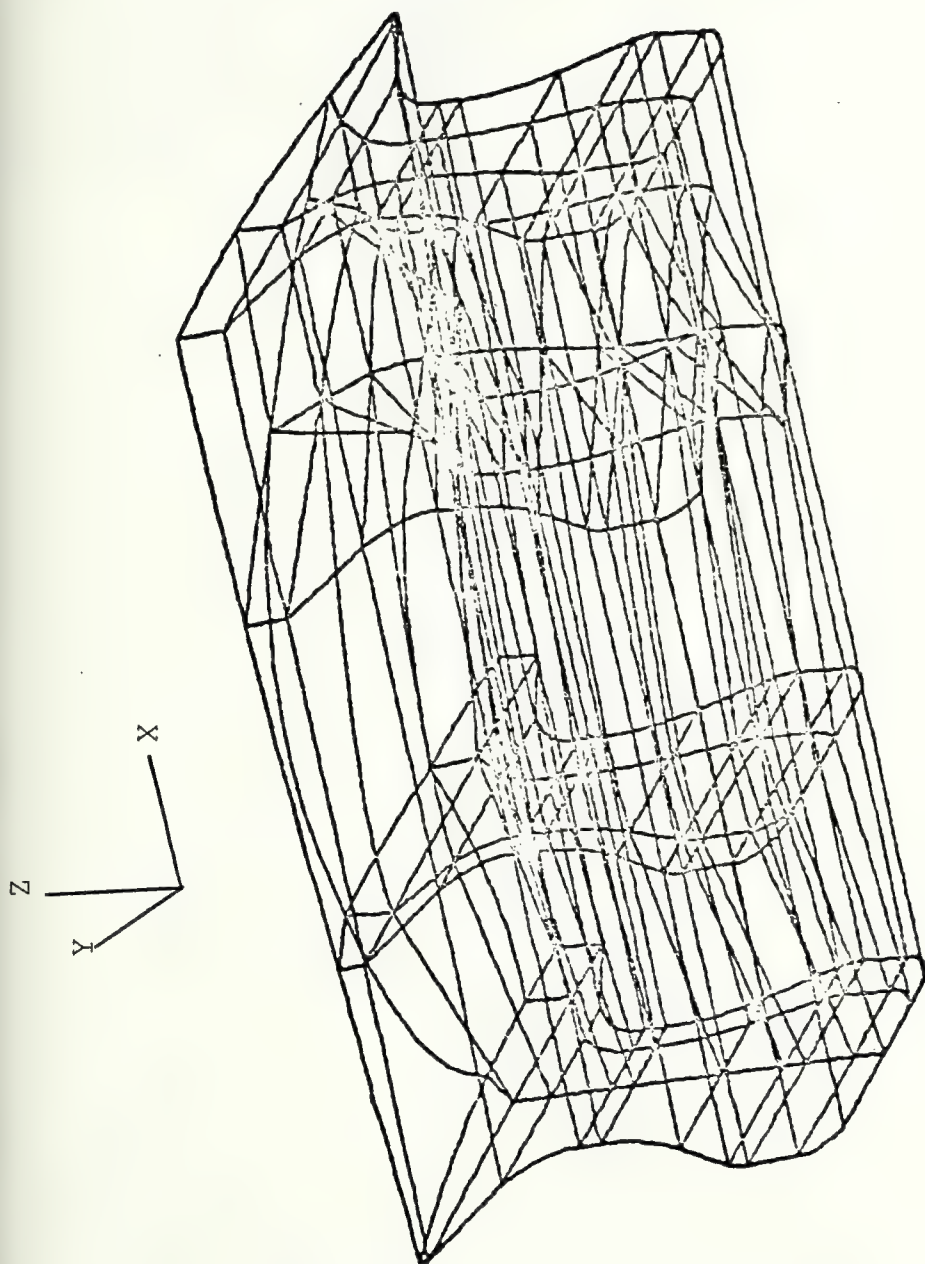
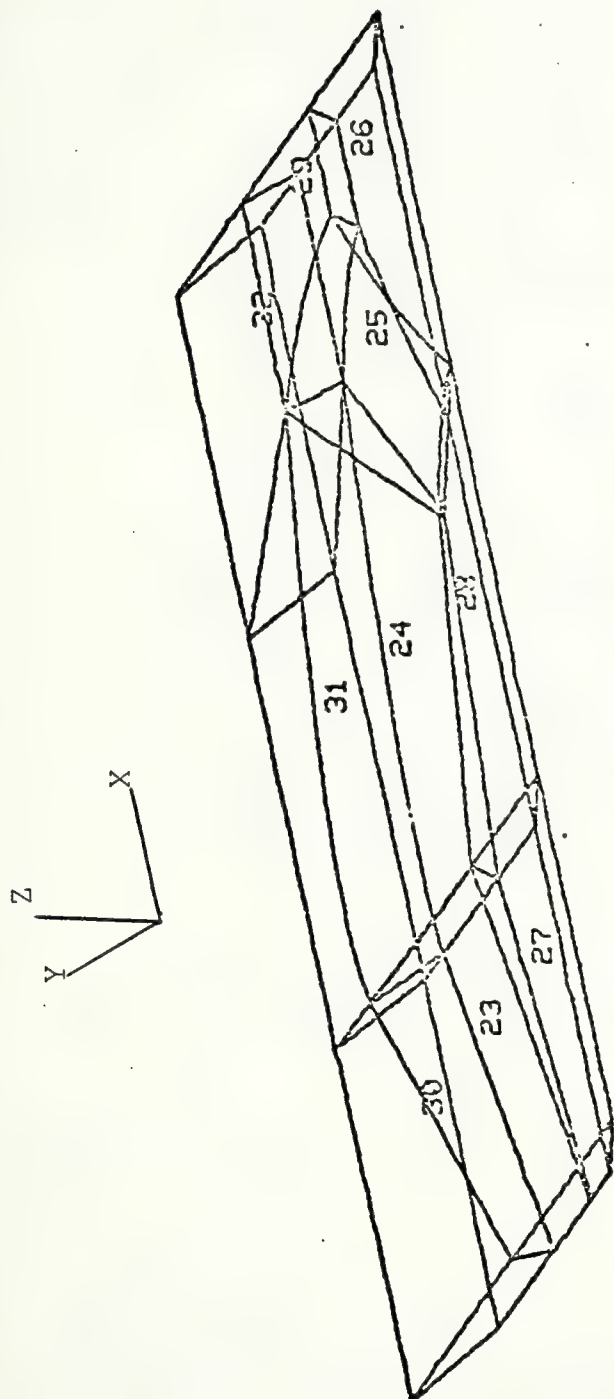


Figure 4. Airfoil Mesh, Elements 1 through 12.



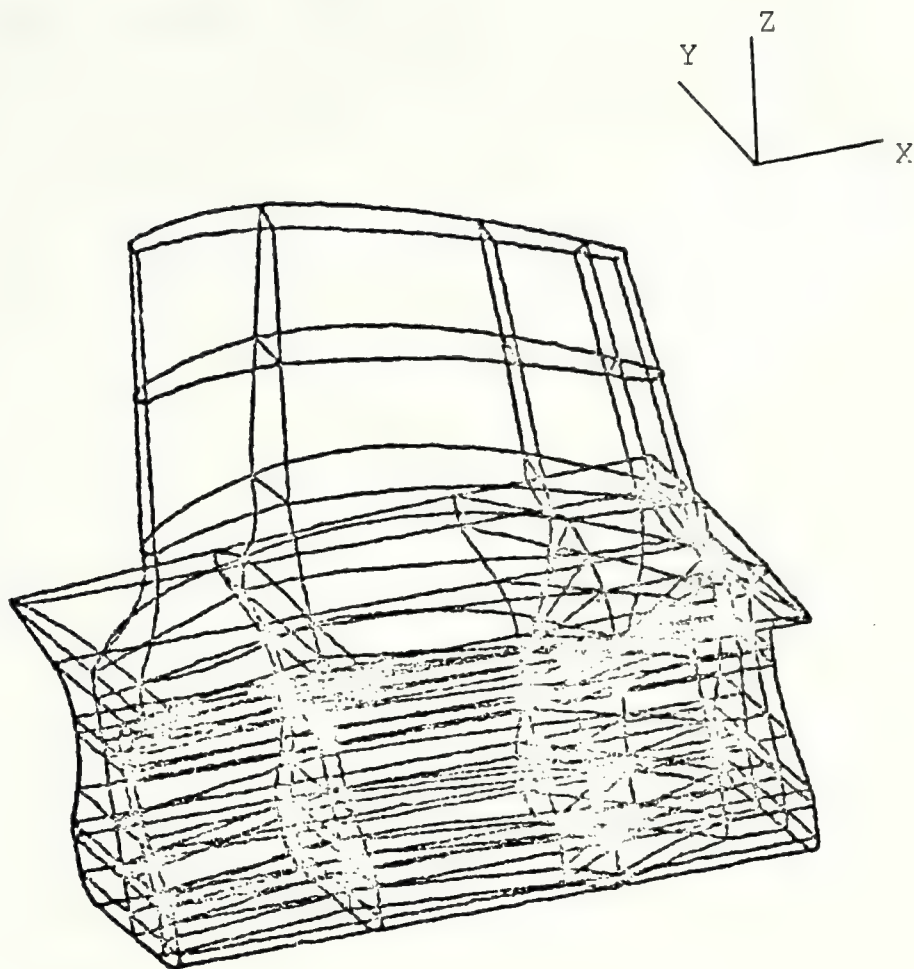
(a) Elements 13 through 102

Figure 5. Attachment Root Mesh,
Page 1 of 2.



(b) Elements 23 through 32

Figure 5. Continued, Page 2 of 2.



EASTERLING-FINITE ELEMENT ANALYSIS
OF A CERAMIC GAS TURBINE BLADE

Figure 6. Completely Assembled Finite Element
Analysis Mesh of Proposed Ceramic Gas
Turbine Blade Design.

III. PRE-PROCESSOR DEVELOPMENT

The ADINA finite element code is a flexible, state-of-the-art code encompassing linear and non-linear, static and dynamic analyses with a choice of six types of elements and twenty material models in various combinations with element types. In this section, an explanation of the facets and capabilities of ADINA used in the blade analysis is given along with a description of the preparations that were necessary prior to execution of an analysis.

A. GENERAL DESCRIPTION OF ADINA

The brittle behavior of the hot-pressed Silicon-Nitride lends itself to the use of the linear isotropic elastic material model. Twenty node isoparametric bricks were used throughout the mesh. ADINA's use of an out-of-core solution scheme and compacted storage of matrices lends itself attractively for analysis of a large mesh which is required for the blade analysis because of the necessity of describing many different geometries.

For the chosen material model and static analysis, the equilibrium equation solved by ADINA is:

$$KU = R$$

where

K = stiffness matrix

U = displacement matrix

R = load matrix

The stiffness matrix is defined as:

$$K = \int_V B^T C B dV$$

where

B = strain-displacement transformation matrix

C = constitutive stress-strain matrix

The elements of B are functions of the natural coordinates r, s, t, being derived from the isoparametric representation of displacements and the inverse of the Jacobian matrix. The integration is carried out in the natural coordinate system of reference, and dV is defined as

$$dV = \det(J) dr ds dt,$$

where det(J) is the determinant of the Jacobian matrix. Integration is accomplished using Gauss quadrature with the resulting matrix elements being stored in compacted form.

With the exception of gravity loading, concentrated nodal forces are the only allowed force inputs incorporated in the ADINA code implemented on the NPS computer. The rotation induced centrifugal body force needed for this analysis was obtained using an external pre-processor to be discussed in Section III-C.

In ADINA, boundary conditions constraining the structure model are imposed by eliminating global degrees of freedom of selected nodes. The real structure was constrained by a Waspaloy disk with a compliant layer interface between the ceramic contact surface and metal disk. Interpretation of

the behavior of this system of materials varies among investigators with consequent variations in modeling the boundary conditions. Without concise information on the behavior of the actual constraining mechanisms and having to limit the scope of this investigation, the tack taken was to eliminate all degrees of freedom for all nodes defining the blade contact surface.

B. MODIFICATIONS TO ADINA

The ADINA code's complex dynamic dimensioning scheme inhibits the user from making major adjustments, additions or deletions to the implemented code. Two modifications, however, were deemed important enough to implement within the code. Other problem related manipulations were implemented by external algorithms which use the ADINA input deck but do not affect the code.

Originally, ADINA came to a STOP statement when a zero or negative determinant of the Jacobian matrix was encountered. This is a valid procedure since the results of subsequent analysis would be incorrect; however, given the inexperience of this investigator and the distortions of various elements due to the requirement of producing a compatible and complete model, numerous elements were suspected of being ill-behaved, and the STOP procedure prevented efficient troubleshooting. The diagnostic output also inhibited efforts to correct model inadequacies because of a lack of identification of the offending node and element. These problems were circumvented

by deletion of the STOP statement and addition to the diagnostic the location of the offending node and element, allowing identification of all improper elements on a single trial run.

ADINA stress output consisted of three normal and three shear stresses for sixteen nodes out of a possible twenty-seven locations for each element being transmitted solely to the line-printer device. The necessity for meaningful manipulation of the massive output derived from the 682 node mesh and the desire for higher density value coverage for input into contour plotting post-processing dictated modification of the ADINA output capabilities. An additional input parameter was defined in the master control cards which allowed the user the option of requesting output values of all twenty-seven possible output locations (Figure 7). A WRITE statement was inserted to output, on a user defined file 57, all nodal stresses. Appendix A details the changes necessary to the ADINA User's Manual [Ref. 1] resulting from these modifications along with other changes from a companion investigator's contributions.

C. CENTRIFUGAL LOADING

Principal modes of loading considered for the blade were thermal, pressure and centrifugal. Given the temperature distribution, ADINA has the capacity for determining thermal gradient induced loads. External pre-processing was chosen for development of pressure and centrifugal loading. The

pressure loading pre-processor was developed by Lieutenant J. Preisel and reported in Ref. 7. A centrifugal loading algorithm was developed by this author and is described herein.

The ADINA version implemented on the Naval Postgraduate School computer accepts only concentrated nodal forces. An algorithm was developed which would calculate the nodal consistent loads equivalent to a rotation induced body force and produce an output of punched cards in a format acceptable to ADINA.

1. Mathematical Formulation

Starting with Newton's Second Law:

$$\underline{F} + \underline{I} = 0$$

$$\underline{I} = -m \underline{a} = -\int_V \rho dV \underline{a}$$

where

\underline{F} = external forces

\underline{I} = inertia forces

m = mass

ρ = mass density

\underline{a} = acceleration

dV = differential volume

A vector of discretized consistent nodal forces (V_i) is desired such that

$$\langle V_i \rangle \{u_i\} = \langle I \rangle \{u\}$$

where

$\langle V_i \rangle$ = vector of consistent loads

$\{u_i\}$ = vector of discrete displacements

$\langle I \rangle$ = vector of inertia force components I_x, I_y, I_z

$\{u\}$ = vector of displacement function components
 $\langle u, v, w \rangle^T$

Using a right-hand cartesian coordinate system and isoparametric element definitions, displacements and coordinates are discretized by

$$\begin{aligned}\{u\} &= \langle h_i \rangle \{u_i\} \\ \{x\} &= \langle h_i \rangle \{x_i\}\end{aligned}\tag{3}$$

where

$\langle h_i \rangle$ = vector of shape functions = $f(r,s,t)$
 $\{x\}$ = continuum coordinates $\langle x,y,z \rangle^T$
 $\{x_i\}$ = discrete nodal coordinates

The acceleration matrix is defined by

$$\underline{a} = \underline{w} \times (\underline{w} \times \underline{r})\tag{4}$$

where

\underline{w} = angular velocity vector = $w_x \underline{i} + w_y \underline{j} + w_z \underline{k}$
 \underline{r} = position vector = $x \underline{i} + y \underline{j} + z \underline{k}$
 \underline{a} = acceleration vector = $a_x \underline{i} + a_y \underline{j} + a_z \underline{k}$

After vector algebra manipulation, one derives

$$\{a\} = [A] \{x^*\}\tag{5}$$

where

$$A = \begin{bmatrix} -(w_y^2 + w_z^2) & w_x w_y & w_x w_z \\ w_x w_y & -(w_z^2 + w_x^2) & w_y w_z \\ w_x w_z & w_y w_z & -(w_x^2 + w_y^2) \end{bmatrix}$$

$$\{a\} = \langle a_x, a_y, a_z \rangle^T$$

$$\{x^*\} = \langle x, y, z \rangle^T$$

Discretizing equations (1) and (5), using equation (3), and substituting into equation (2), one has

$$\langle V_i \rangle \{u_i\} = \rho \int \langle x_i \rangle \{h_i\} [A] \langle h_i \rangle dVol \{u_i\}$$

where

$$dVol = \det J \, dr \, ds \, dt$$

Simplifying and rearranging yield the following equation which was used to program the computer algorithm for development of consistent loads:

$$\{V_i\} = \rho \int \{h_i\} [A] \langle h_i \rangle \{x_i\} \, dVol$$

2. Computer Algorithm for Program Centrifugal Load

The centrifugal load algorithm is generalized for use by any ADINA three-dimensional brick element mesh defined by eight to twenty nodes per element. A dynamic dimensioning scheme is used in order to conserve core and simplify changes dictated by the size of the user's mesh.

a. Input

Angular velocity about the three cartesian axes, mass density and an ADINA input deck are the required input for calculations in Program Centrifugal Load. In addition, several housekeeping parameters are required as defined in Appendix C.

The ADINA input deck provides node coordinates and mesh connectivity. By using the subroutine RDADIN, duplication of the considerable effort to punch and debug a large mesh is obviated. This subroutine should prove very useful for future users of ADINA who wish to manipulate mesh values.

b. Calculation of Consistent Nodal Loads

The user can select two to six Gauss point integration in the calculations of the consistent loads. Shape functions were derived from Ref. 8, which are the same used in ADINA. After calculation of loads for each element, the contributions to each node are summed in each of the three cartesian coordinate directions to yield the desired output of concentrated nodal forces. ADINA has the capability of summing concentrated nodal forces in a common direction for a given node; therefore, loads other than centrifugally induced may be input by the user without necessitating modification to Program Centrifugal Load output.

c. Output

Four options allow the user to restrict the output to what is desired for the particular problem under investigation:

(1) ICCHK1. This flag allows the option for printout of the nodal coordinate data and mesh connectivity. Since this data is also output by use of the ADINA code and PSAP1, the user would normally not desire this option from Program Centrifugal Load.

(2) ICCHK2. This parameter controls the option for controlling the output of consistent loads calculated for each element. The information may be useful in comparing the contributions of various elements.

(3) ICCHK3. The information required by ADINA is controlled by this parameter which allows printout of the totaled consistent loads for each node for the three coordinate directions.

(4) ICCHK4. This parameter gives the user the option of not punching the data cards as may be the choice for a first run.

Other output items include the total force in each of the three coordinate directions and the total number of load cards punched. The latter value is required by ADINA as parameter NLOAD on the Load Control Card described in Section IV of Ref. 1.

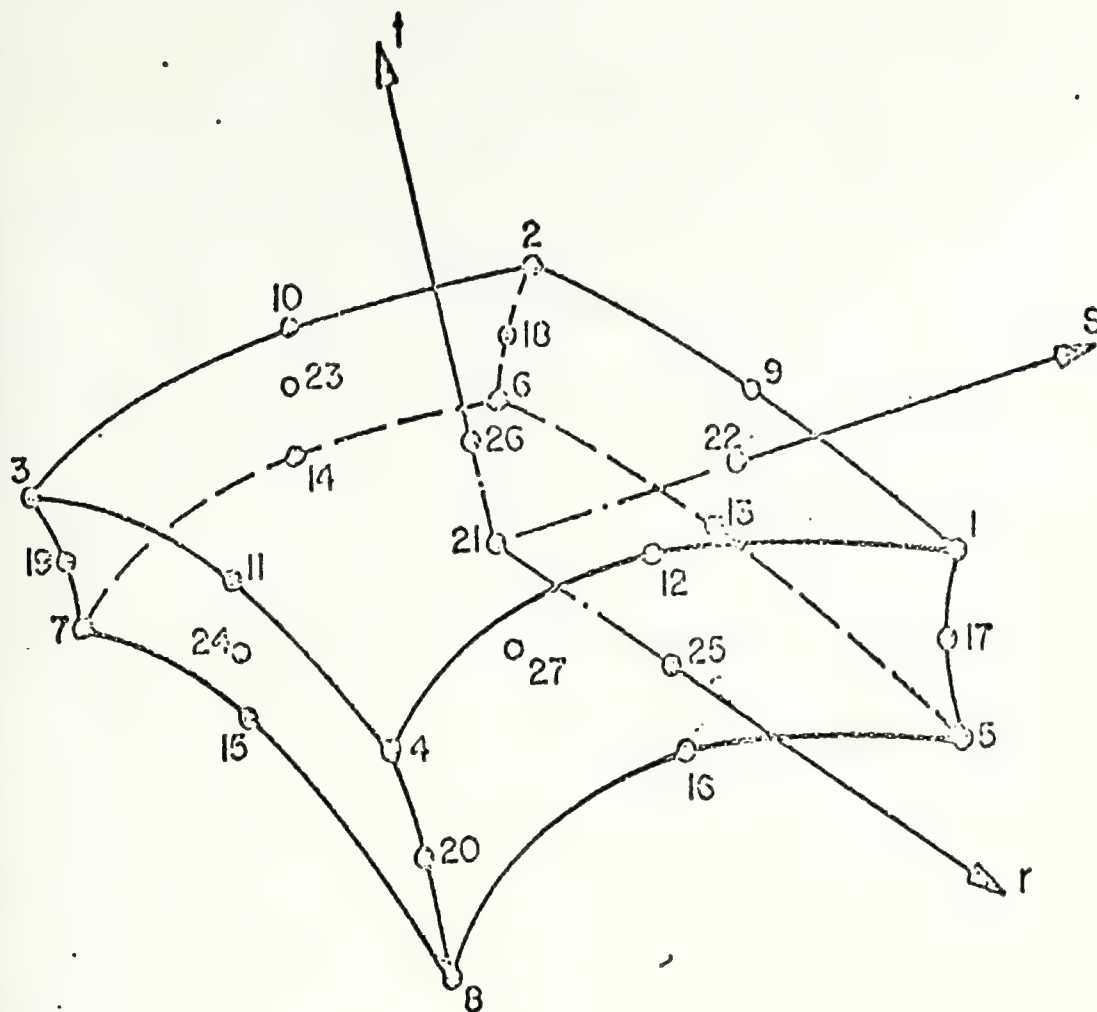


Figure 7. Stress Output Locations for ADINA Linear Static Analysis Results.

IV. PROBLEM SOLUTION

The ADINA problem solution was executed for two, three and four Gauss points used for development of the stiffness matrix. Static linear analysis was chosen for the solution method, and a material model for isotropic behavior was used. A Young's modulus of 4.0×10^7 psi was used, corresponding to the approximate value for hot-pressed Silicon-Nitride at 2500°F. Poissons ratio was chosen to be 0.33. Loading was accomplished using concentrated nodal forces with values obtained from the centrifugal load pre-processor for an angular velocity of 42,000 rpm about the X-axis. Table 1 summarizes the execution times for the three analyses.

OPERATION	ORDER OF INTEGRATION		
	2	3	4
INPUT PHASE	22.74	20.83	24.52
MATRICES ASSEMBLAGE	717.19	1437.83	3305.98
TRIANGULARIZATION OF STIFFNESS MATRIX	637.36	663.63	705.98
STEP BY STEP SOLUTION	110.78	111.29	110.11
TOTAL SOLUTION TIME	1549.57	2234.96	4148.07

TABLE 1. ADINA PROBLEM SOLUTION TIMES (seconds)

V. POST-PROCESSOR DEVELOPMENT

Static linear analysis using the modified version of ADINA yields three orthogonal displacements for each input node and three normal and three shear stresses at each of twenty-seven locations in a twenty node brick. The value of the stress components calculated at a given node will, in general, differ from element to element. Experience has shown that the straight average of these values give the best estimate of the real stress at that node [Ref. 8]. For problems involving brittle material, the principal stresses (particularly the maximum principal stresses) are the primary values of concern in predicting failure of the modeled component. These factors necessitated the development of post-processors to manipulate the ADINA output into the values required for analysis of results. The consequent output of these post-processors was still unmanageable from the standpoint of visual perception of the relation of the results with model geometry. Additional post-processors were, therefore, developed to yield the coordinates and connectivity of a twenty-seven node brick mesh and output of punched cards in a format applicable to the contour plotting code implemented on the Naval Postgraduate School computer.

A. PROGRAM KCONT

Prior to data manipulation, the coordinates of all nodes of the twenty-seven node brick mesh and the connectivity must be known. Program KCONT was developed to accomplish this task by using the basic isoparametric finite element model equalities:

$$x = \sum_{i=1}^n h_i x_i$$

$$y = \sum_{i=1}^n h_i y_i$$

$$z = \sum_{i=1}^n h_i z_i$$

where

x, y, z = global coordinates of any point
within the element

x_i, y_i, z_i = global coordinates of the nodes
defining the element

h_i = the array of shape factors as
the function of the natural
coordinates r, s, t .

Program KCONT first reads an ADINA input deck. It takes the input node coordinates and mesh connectivity data and computes the coordinates of the additional nodes. The twenty-seven node brick mesh connectivity is assembled and is output along with the coordinates of all nodes to the line printer and to the user defined storage files in accordance with the instructions in Appendix D.

B. PROGRAM STRESS

Program Stress reads the file defined by the ADINA code user for storage of stress results and the files storing the total connectivity and coordinate information, averages all values for each given node and determines the principal stresses from the average values. The output consists of storage of the results on user defined files and a line printer presentation. Refer to Appendix E for instructions on using Program Stress.

C. PROGRAM CONTOUR PLOT DATA

Data required by the two-dimensional iso-line contour plotting code consists of housekeeping information in the form of defined NAMELISTS, coordinate data of nodes on the analysis plane, four-node two-dimensional connectivity and input values of information to be plotted. Program Contour Plot Data was designed to define the desired analysis plane(s) and yield a punched data deck of coordinates and plotting value information. A graphics generated plot of node points and global numbers is also generated to assist the user in the formulation of the connectivity. The connectivity information must then be punched on data cards by the user and inserted into the appropriate data deck location.

Analysis planes can be defined with any of three options. The user can simply input the nodes to be used on data cards in the format I6I5. Most models, however, have surfaces of interest which are parallel to one of the three orthogonal

planes defined by the axes of the right-handed cartesian coordinate systems. Should this be the case, the user may define bounding values of a coordinate for testing the mesh for the desired nodes defining the analysis plane. If the user inputs both bounding values equal, an equality test is made in order to define the desired plane.

The program as written takes the array of plotting values from the file containing the principal stresses established by Program Stress and uses the maximum principal stress array (variable SIGMX) for plotting contours. A user may redefine the read statement (line CTRP2470 of Appendix F) in order to input the desired values for a particular problem, filling the array SIGMX with the desired plotting data. Coordinate values are read from file 58 established by Program KCONT.

The resulting punched deck of cards requires the input of connectivity and NAMELISTS &OPTION and &PICT in locations designated by Ref. 3.

D. PROGRAM CONTOUR PLOT

The contour plotting routine chosen to display stress results was developed by Gary L. Giles of the Langley Research Center for use with a variety of graphic systems including the CALCOMP plotter installed at the Naval Postgraduate School. Unfortunately, the NPS graphics software package rotates the plotting axis 90° clockwise from the original software conception of plot orientation. In order

to achieve optimum utilization of the plotting surface, the decision was made to modify the contour plot routine to utilize the NPS graphics package but yield a plot oriented with the horizontal plotting axis along the length of the paper roll as originally conceived by the CALCOMP manufacturer. In general this requires inputting to the drawing routines of NPS the negative of the desired vertical coordinate in the calling location for the x-coordinate and the positive horizontal coordinate in the calling position of the y-coordinate.

The plotting origin for the analysis plane is chosen to be the geometric center. In order to place the plot properly on the plotting surface for any set of coordinates, parameters PXORGN and PYORGN were added to NAMELIST &OPTION with default values of 0.0. Inputting the user's coordinates for the center of his analysis plane properly centers the plot on the plotting surface.

VI. RESULTS OF ADINA STRESS ANALYSIS

Stress results were analyzed using a program developed to compare the results of the two, three and four point Gauss integration analyses and plotting the iso-maximum principal stress contours for various chosen planes of analysis. Severe surface tensile stress concentrations were observed in the region immediately above the ceramic-disk contact region.

A. COMPARISON OF MAXIMUM PRINCIPAL STRESSES

Formulation of the stiffness matrix using two-point integration resulted in higher stresses caused by a lower order in integration, creating a more flexible system which concurs with finite element research [Ref. 8]. Some investigators have found that the results of reduced order integration reflect more accurately the actual component stresses because the strict mathematical development of the finite element method creates a system which is stiffer than the actual component. Insufficient experimental data, uncertainty of boundary conditions and the lack of an adequate convergence study prevent a statement of qualitative opinion of the actual stresses involved for the blade analysis; however, noting the failure rate of the few specimens which have been tested, it is probable that tensile stresses in the dovetail are higher than predicted.

Table 2 illustrates the differences between the three analyses by comparing the greatest maximum principal stress

encountered, the average of maximum principal stresses and the Euclidean norm of the differences between results for different integration order analysis.

	ORDER OF INTEGRATION		
	2	3	4
MAXIMUM PRINCIPAL STRESS	52862.39	46671.73	4674.73
AVERAGE MAX PRINCIPAL STRESS	6198.79	6100.42	6093.98
INTEGRATION ORDER ANALYSES COMPARED			
	2-3	2-4	3-4
EUCLIDEAN NORM	122.2901	122.6712	4.7777

TABLE 2. COMPARISON OF RESULTS OF ANALYSES USING TWO, THREE AND FOUR POINT INTEGRATION (psi)

The maximum value occurred at node 394 which is located immediately above the contact region at level 2.9029 (Figure 8). Results from the three and four order of integration analyses showed little difference, indicating that the order of integration necessary for the "exact" evaluation of the stiffness matrix elements is being approached.

B. CONTOUR PLOT ANALYSIS

Thirty-four analysis planes were defined for contour plots. Presented herein are eight of these planes which show the regions of highest stress concentrations and the distribution of stresses in the airfoil. Figure 9 illustrates the relative position of the six plots of analysis

planes in the attachment root. The level number refers to the z-coordinate value of the nodes on that plane. The figures presented consist of an element plot, illustrating the coverage of input values to the contour plotting code, and the iso-stress plots from the second and fourth order integration analyses.

Figure 10 shows severe surface stress concentrations on both pressure and suction sides of the dovetail in the region immediately above the blade-disk contact surface. This result is consistent with other analyses of blades of similar design regardless of the boundary conditions used and loading scheme, indicating the stress distribution is principally a function of geometry. The orders of magnitude of the stress concentrations are fifty ksi for the two-point integration plot and forty-five ksi for the four-point integration plot.

Figures 11 and 12 present another view of the vertical stress distribution in dovetail. Noteworthy information from these plots is that the stress concentration is generally uniform along the length of the attachment root.

Figures 13 through 15 are plots of three horizontal surfaces in the region of high stress concentration in the dovetail.

Another region of concern in the design of a ceramic turbine blade was the fillet area of the airfoil. Figures 16 and 17 illustrate the stress distributions in the airfoil as viewed from the pressure and suction sides. Some stress

concentration does occur at the leading and trailing edges at the mid-fillet height; however, the values are minor compared to magnitudes obtained in the attachment root and relative to the strength of the material. Test results to date also indicate the adequacy of the fillet design as no failures have originated in this region except for impact initiated failure caused by flying debris upon failure of other blades.

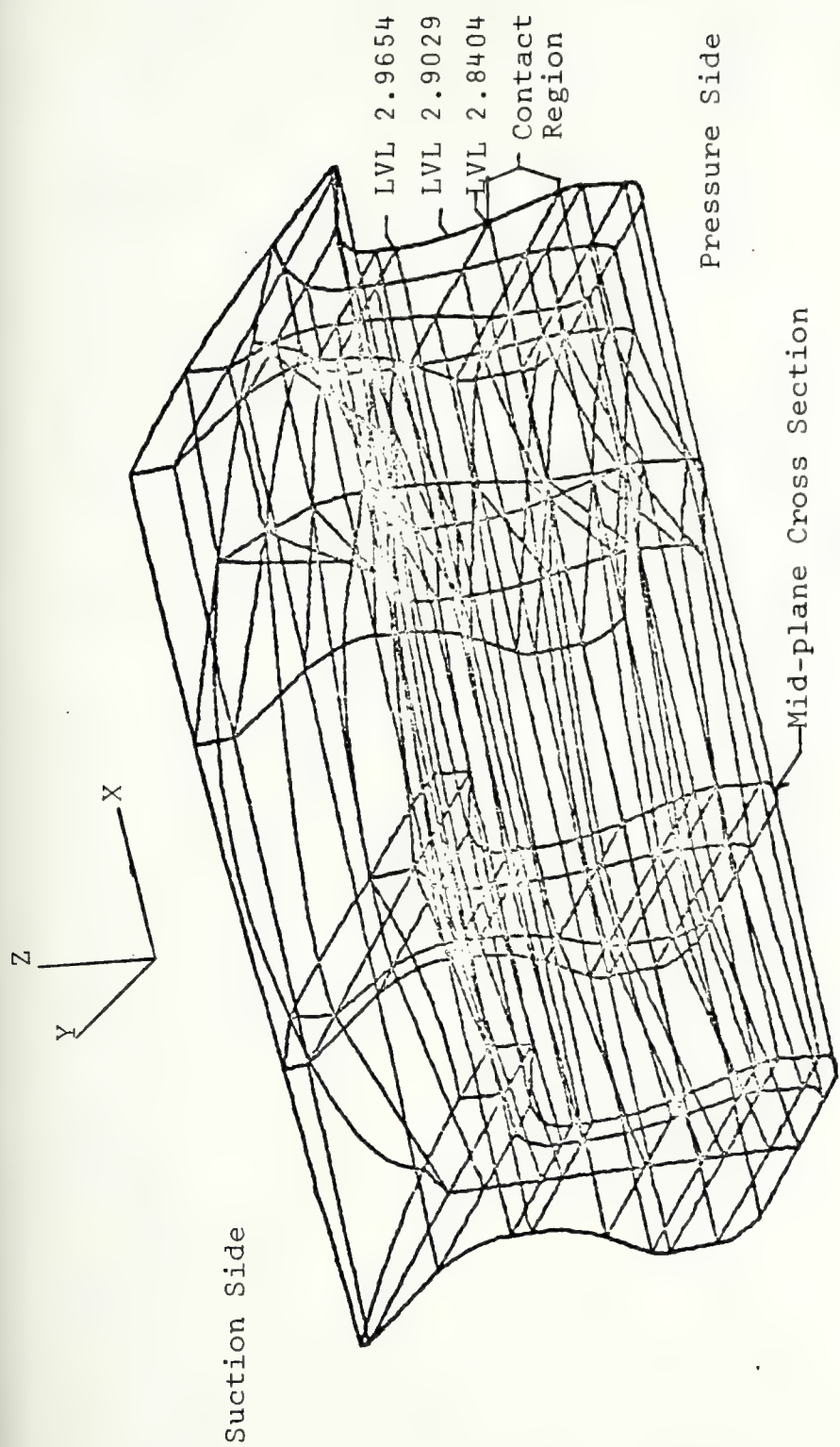
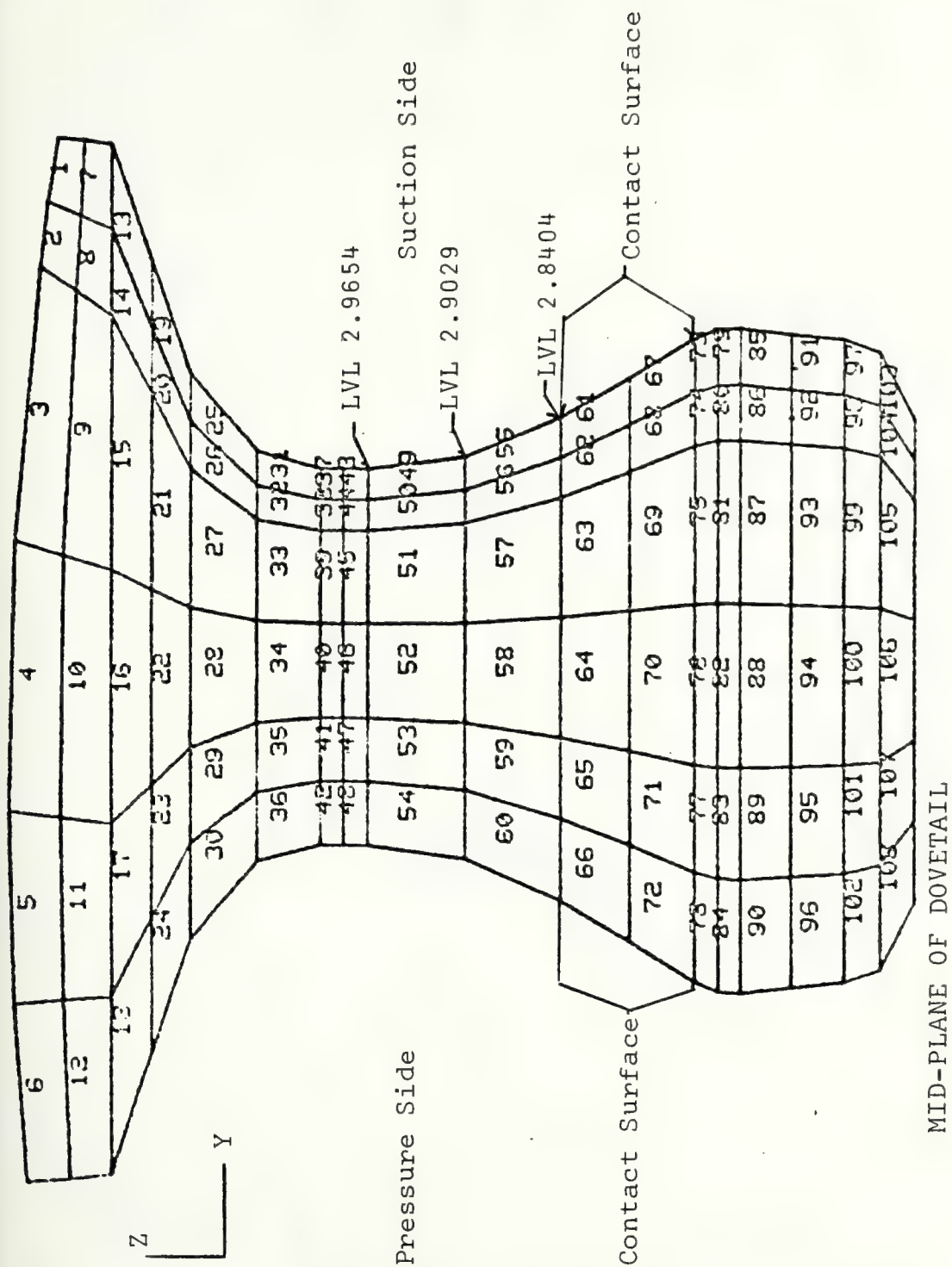
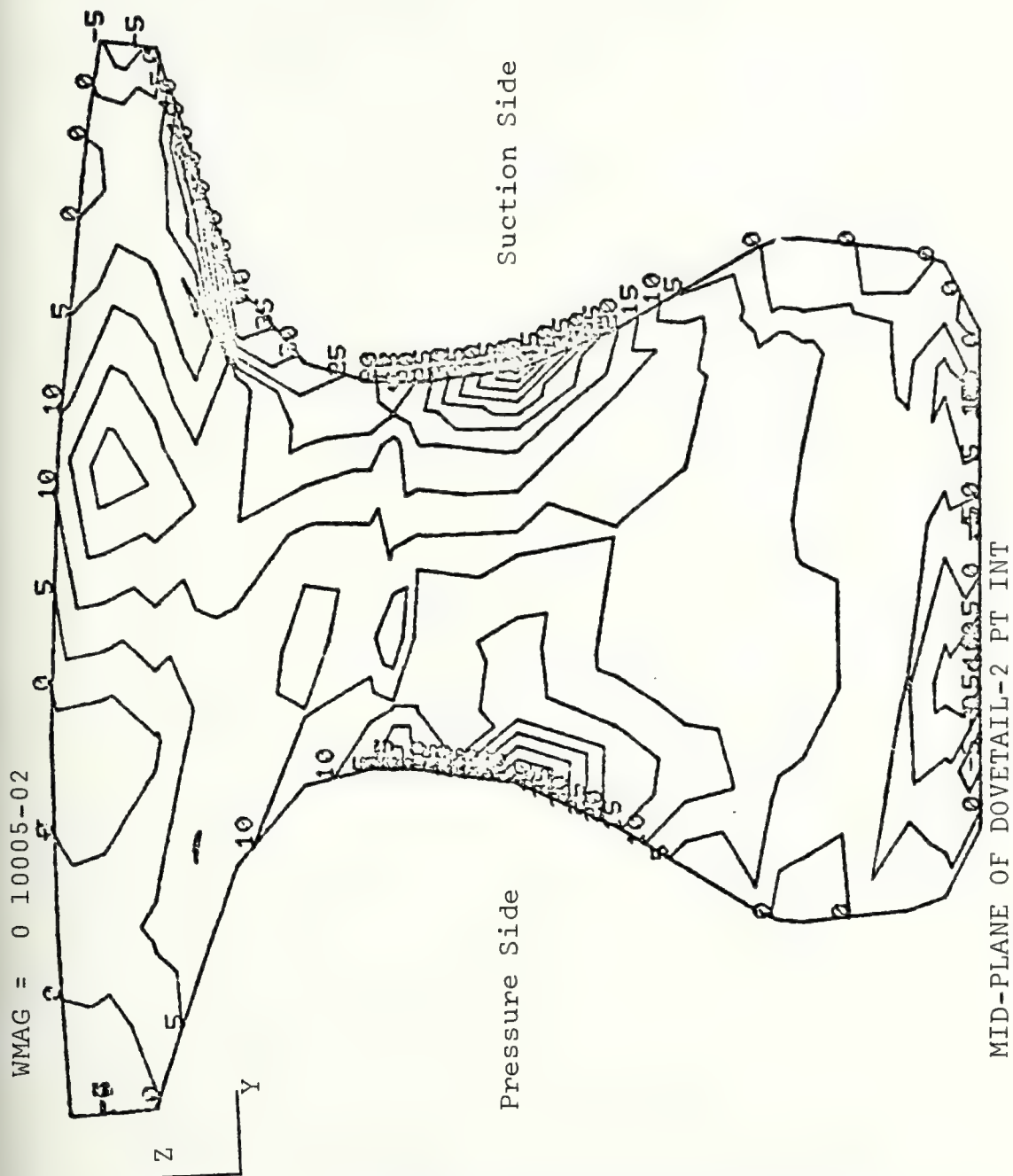


Figure 9. Attachment Root Finite Element Mesh.



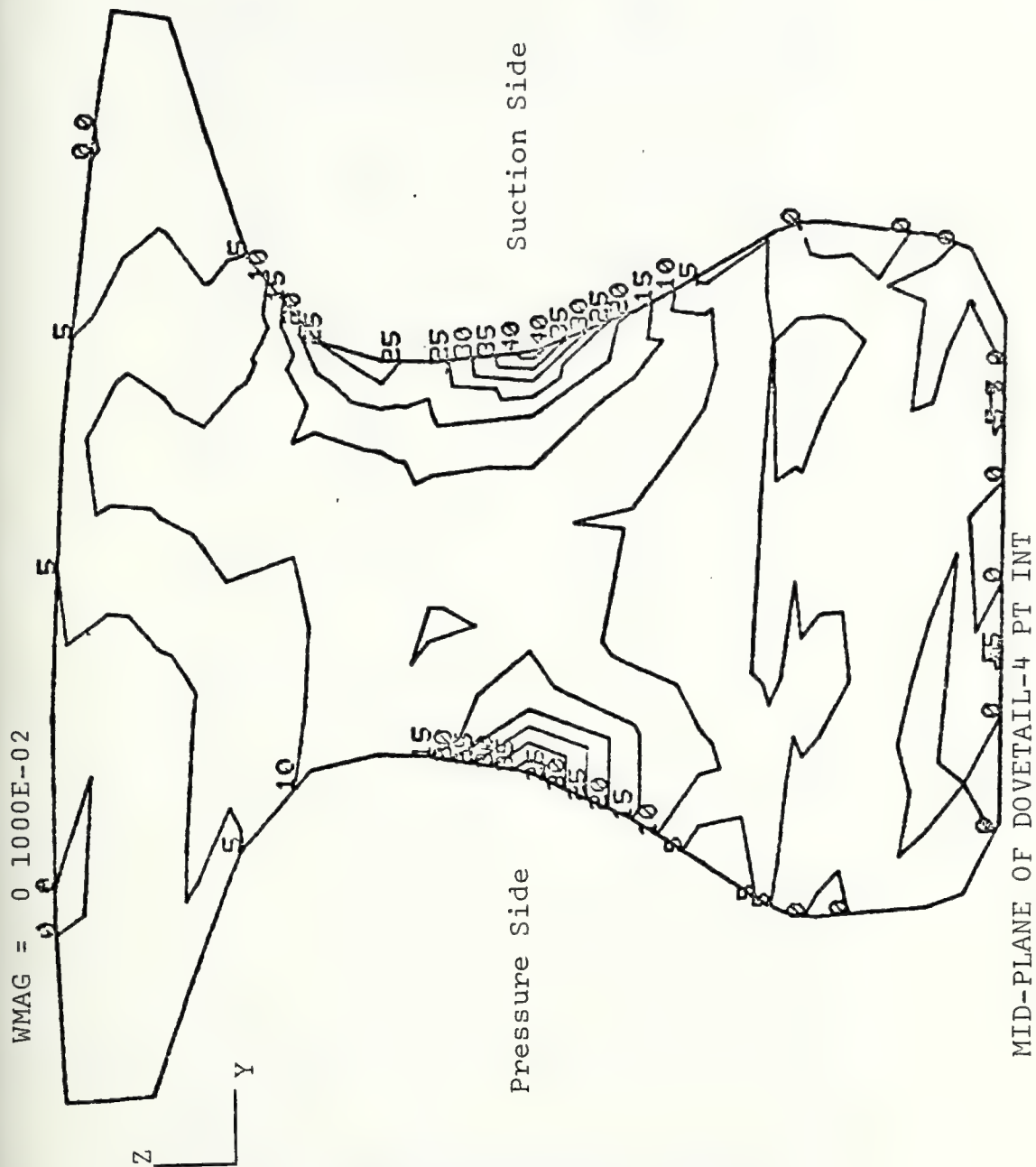
(a) 2-D Element Arrangement

Figure 10. Mid-Plane Dovetail Cross-Section,
Page 1 of 3.



(a) Maximum Principal Stress Iso-lines for 2-pt Order Integration (ksi)

Figure 10. Continued, Page 2 of 3.



(b) Maximum Principal Stress Isolines for 4-pt Order Integration (ksi)

Figure 10. Continued, Page 3 of 3.

A: Level 2.9654
 B: Level 2.9029
 C: Level 2.8404
 D: Contact Surface



	2	3	4	5	6
7	8	9	10	11	12
13	14	15	16	17	18
19	20	21	22	23	24
25	26	27	28	29	30
31	32	33	34	35	36
37	38	39	40	41	42
43	44	45	46	47	48
49	50	51	52	53	54
55	56	57	58	59	60
61	62	63	64	65	66
67	68	69	70	71	72
73	74	75	76	77	78
79	80	81	82	83	84
85	86	87	88	89	90
91	92	93	94	95	96
97	98	99	100	101	102
103	104	105	106	107	108

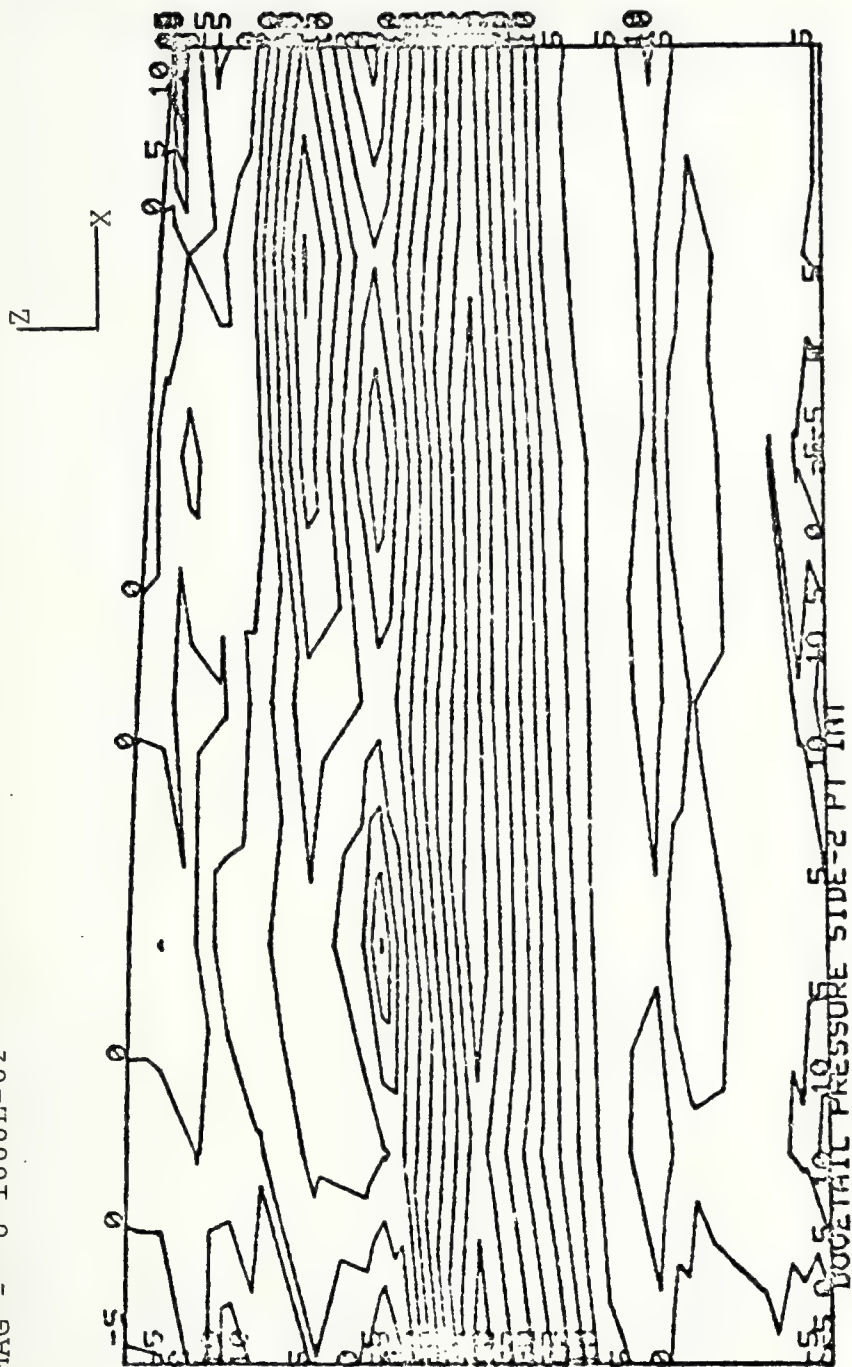
A
 D
 C
 D

DOVETAIL PRESSURE SIDE

(a) 2-D Element Arrangement

Figure 11. Dovetail Pressure Side,
 Page 1 of 3.

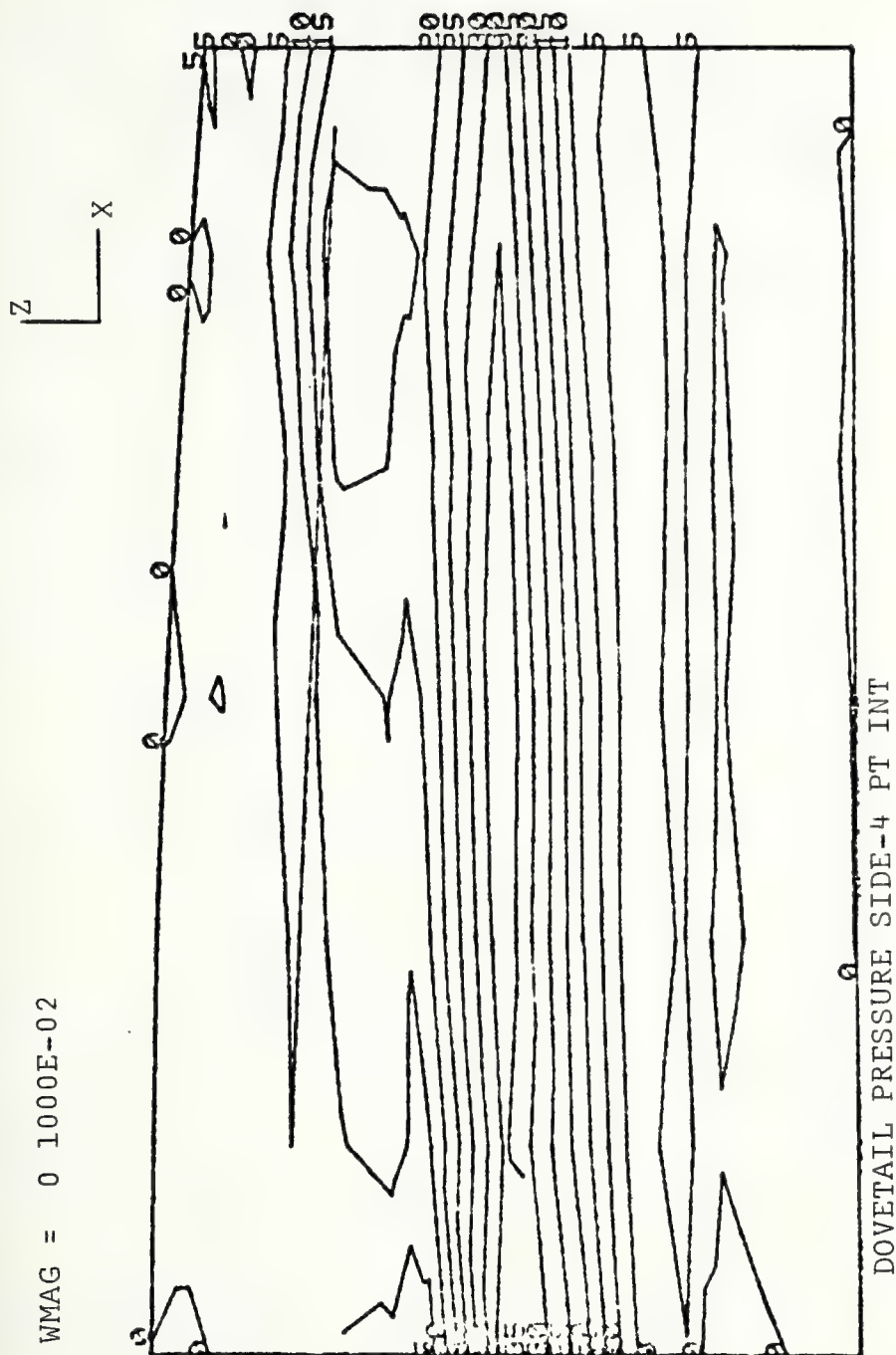
WMAG = 0 1000E-02



(b) Maximum Principal Stress Iso-lines
for 2-pt Order Integration (ksi)

Figure 11. Continued, Page 2 of 3.

WMAG = 0 1000E-02



(c) Maximum Principal Stress Iso-lines
for 4-pt Integration Order (ksi)

Figure 11. Continued, Page 3 of 3.

Z
X

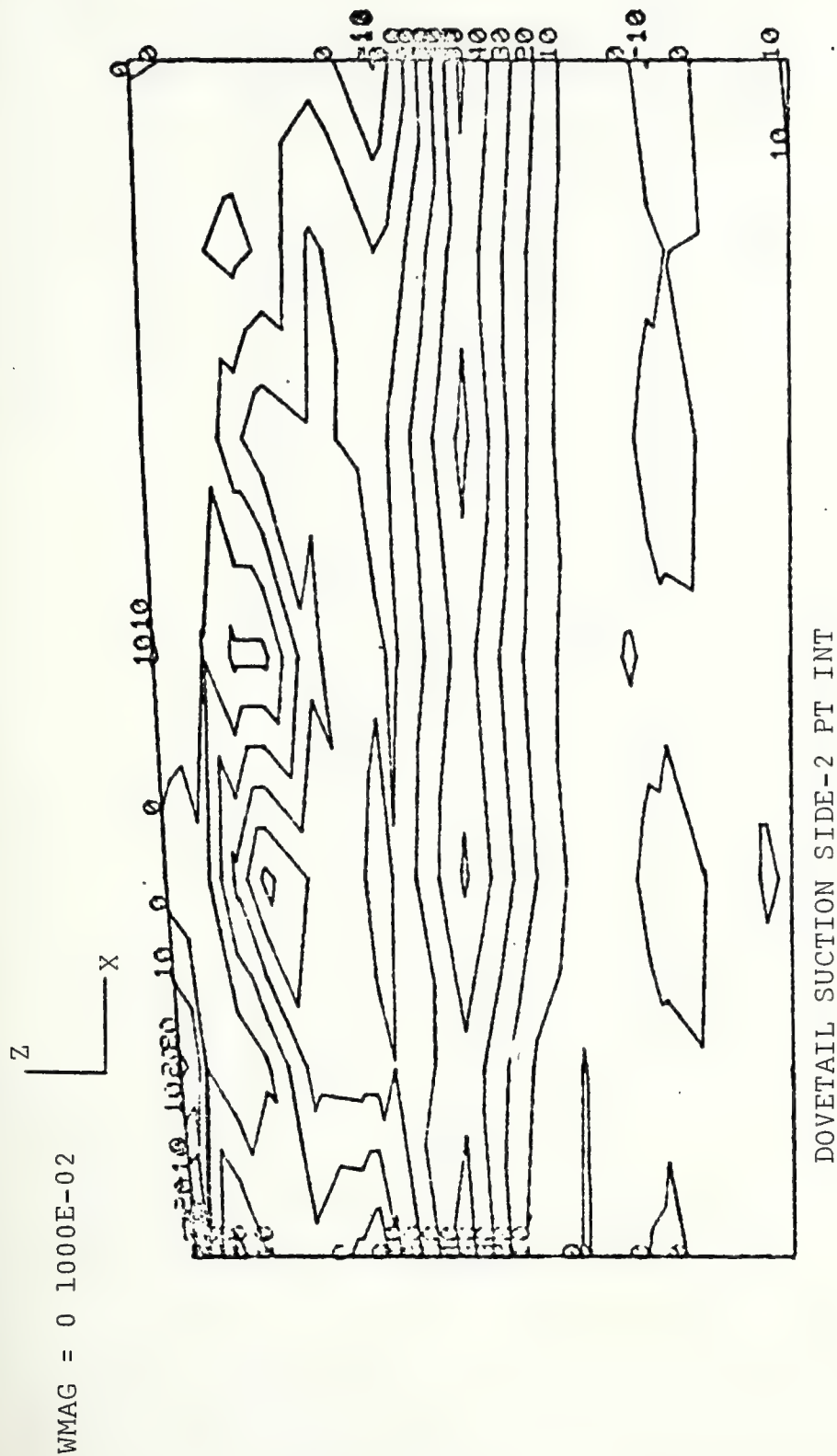
A: Level 2.9654
B: Level 2.9029
C: Level 2.8404
D: Contact Surface

1	2	3	4	5	6
7	8	9	10	11	12
13	14	15	16	17	18
19	20	21	22	23	24
25	26	27	28	29	30
31	32	33	34	35	36
37	38	39	40	41	42
43	44	45	46	47	48
49	50	51	52	53	54
55	56	57	58	59	60
61	62	63	64	65	66
67	68	69	70	71	72
73	74	75	76	77	78
79	80	81	82	83	84
85	86	87	88	89	90
91	92	93	94	95	96
97	98	99	100	101	102
103	104	105	106	107	108

DOVETAIL SUCTION SIDE

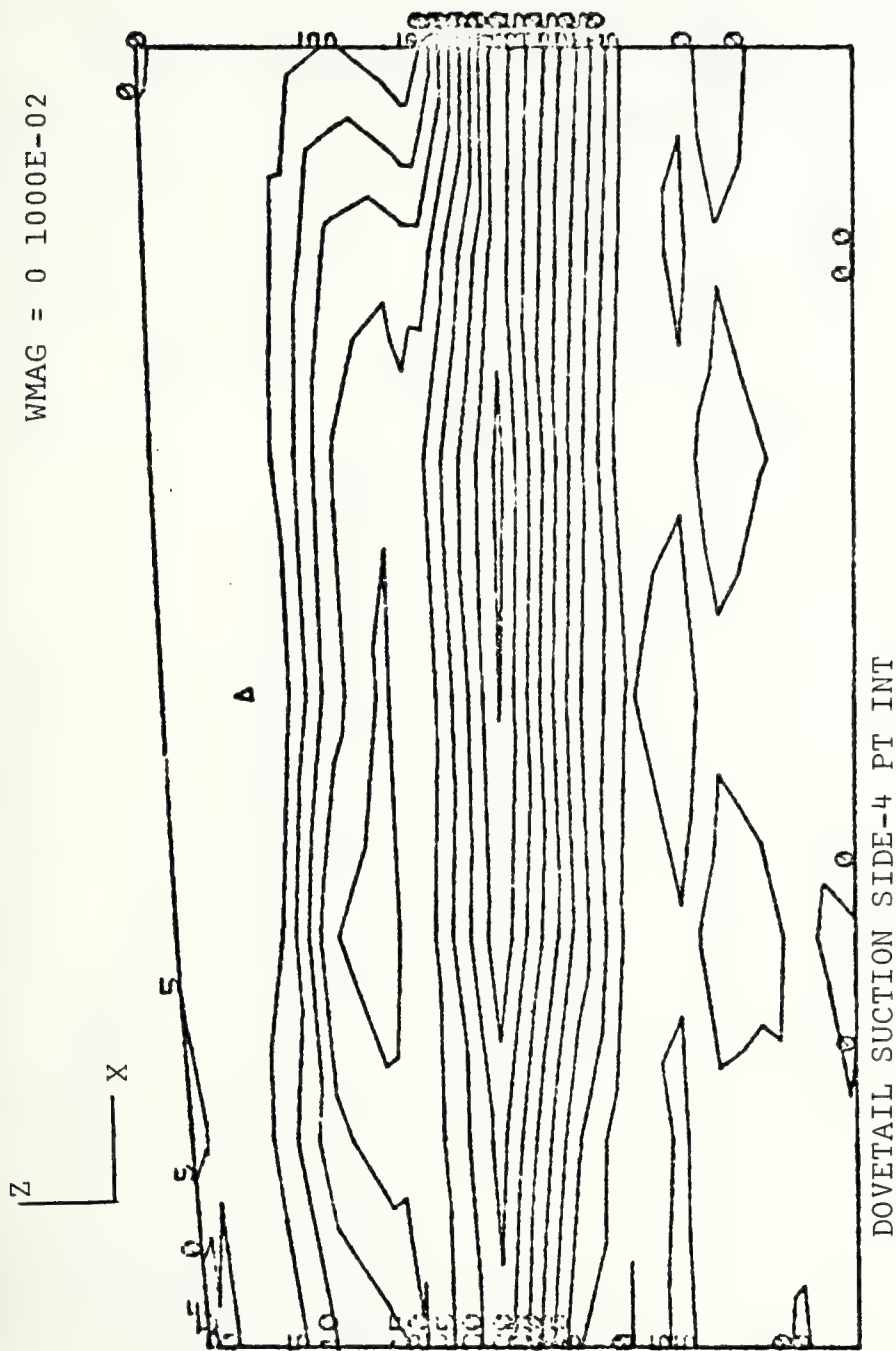
(a) 2-D Element Arrangement

Figure 12. Dovetail Suction Side,
Page 1 of 3.



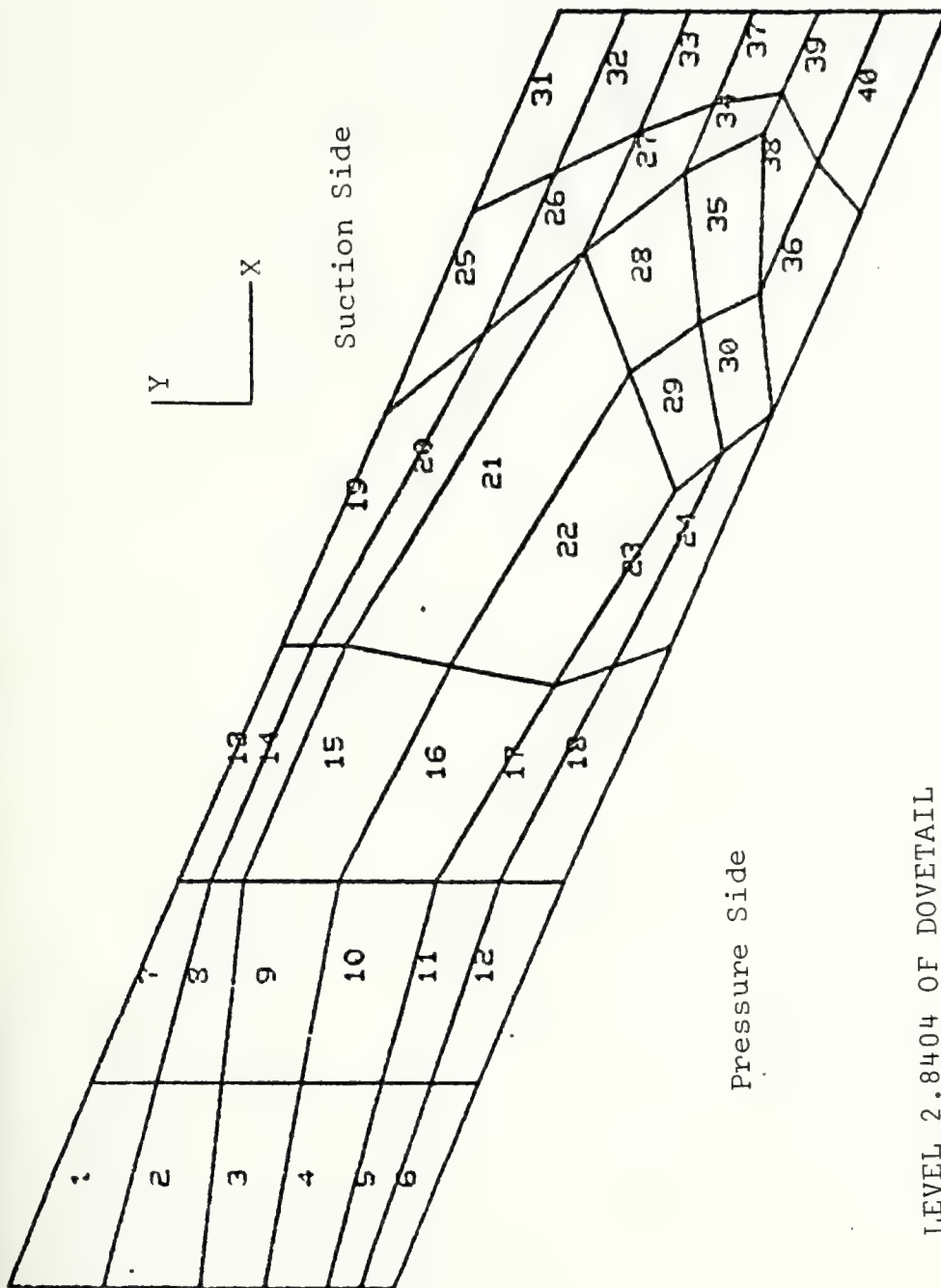
(b) Maximum Principal Stress Iso-lines
for 2-pt Integration Order (ksi)

Figure 12. Continued, Page 2 of 3.



(c) Maximum Principal Stress Iso-lines
for 4-pt Integration Order (ksi)

Figure 12. Continued, Page 3 of 3.

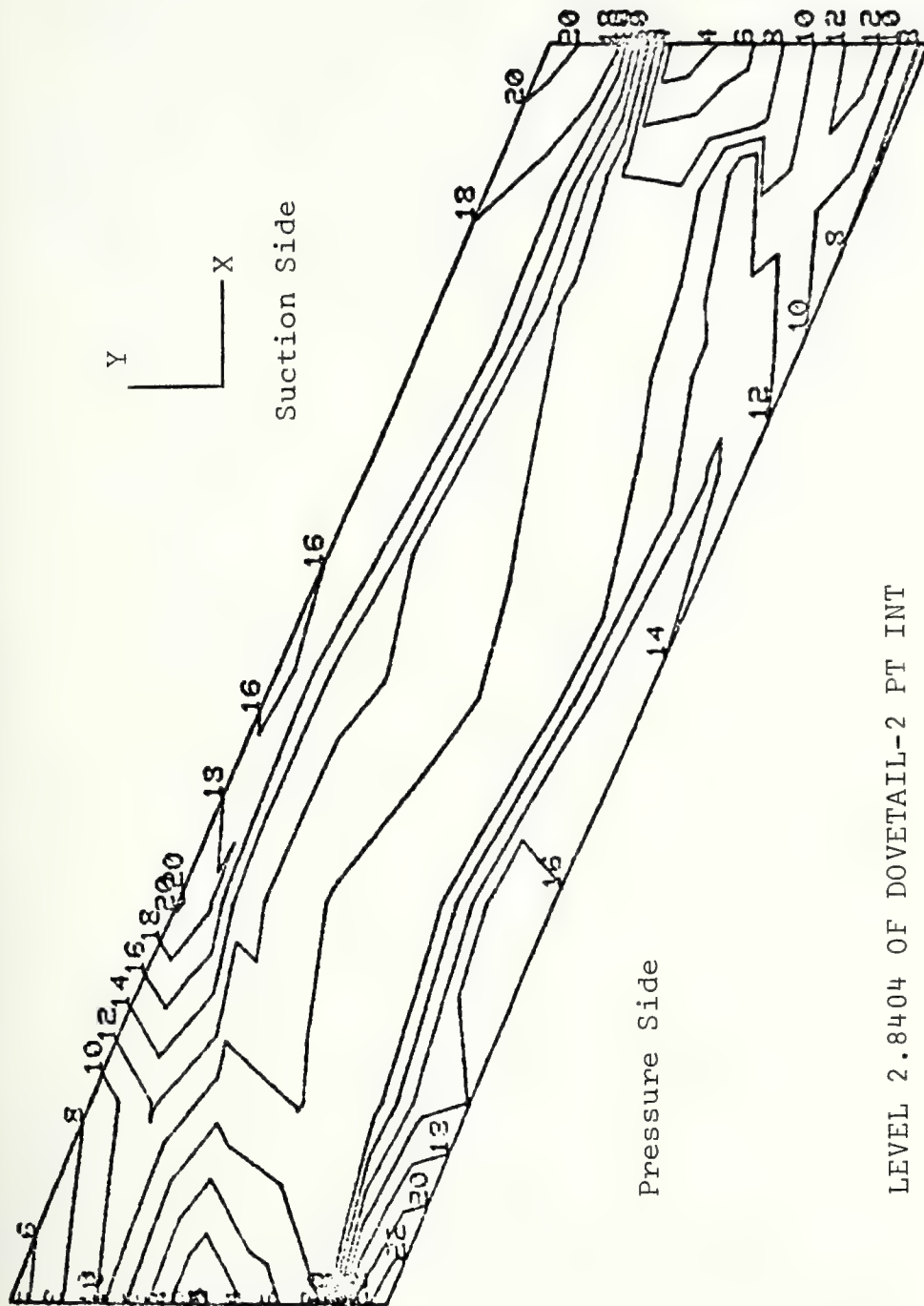


LEVEL 2.8404 OF DOVETAIL

(a) 2-D Element Arrangement

Figure 13. Level 2.8404, Page 1 of 3.

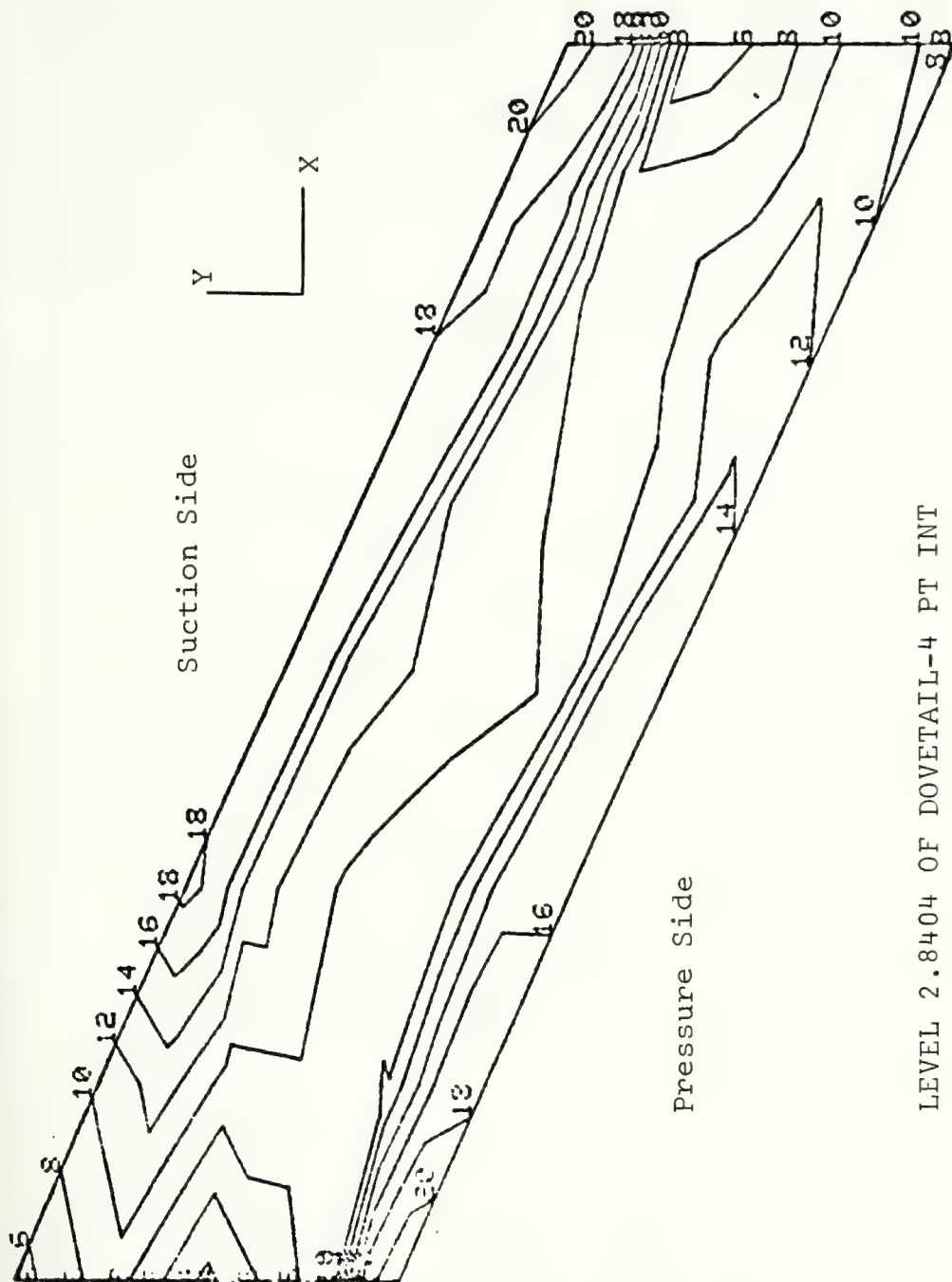
WMAG = 0 1000E-02



LEVEL 2.8404 OF DOVETAIL-2 PT INT

(b) Maximum Principal Stress Iso-lines
for 2-pt Integration Order (ksi)

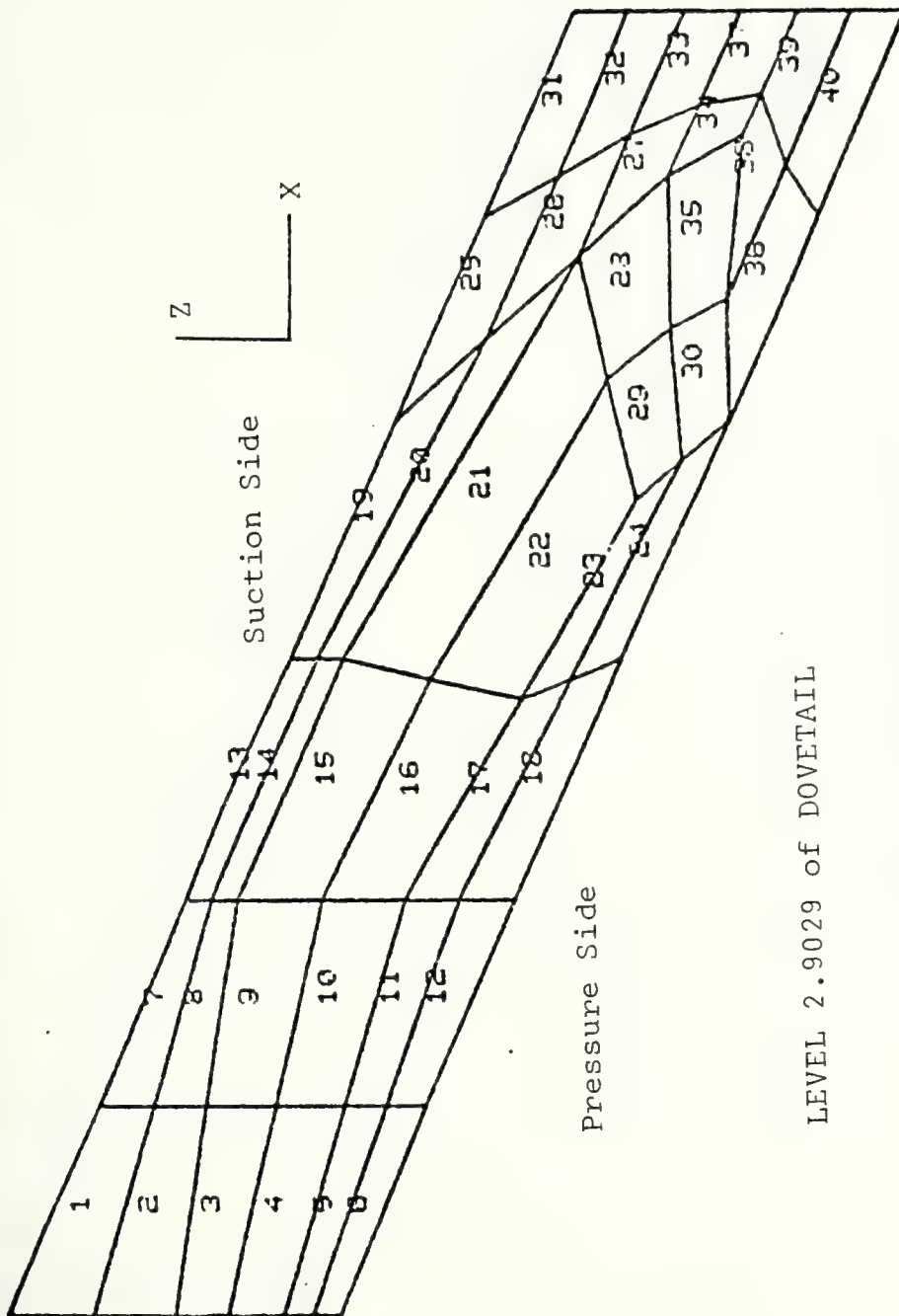
Figure 13. Continued, Page 2 of 3.



LEVEL 2.8404 OF DOVETAIL-4 PT INT

(c) Maximum Principal Stress Iso-lines
for 4-pt Integration Order (ksi)

Figure 13. Continued, Page 3 of 3.



LEVEL 2.9029 of DOVETAIL

(a) 2-D Element Arrangement

Figure 14. Level 2.9029, Page 1 of 3.

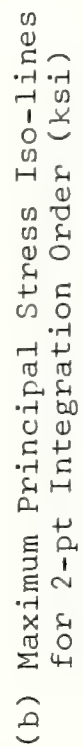


Figure 14. Continued, Page 2 of 3.

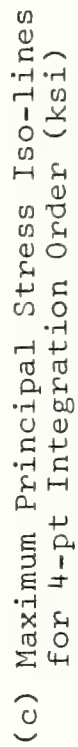
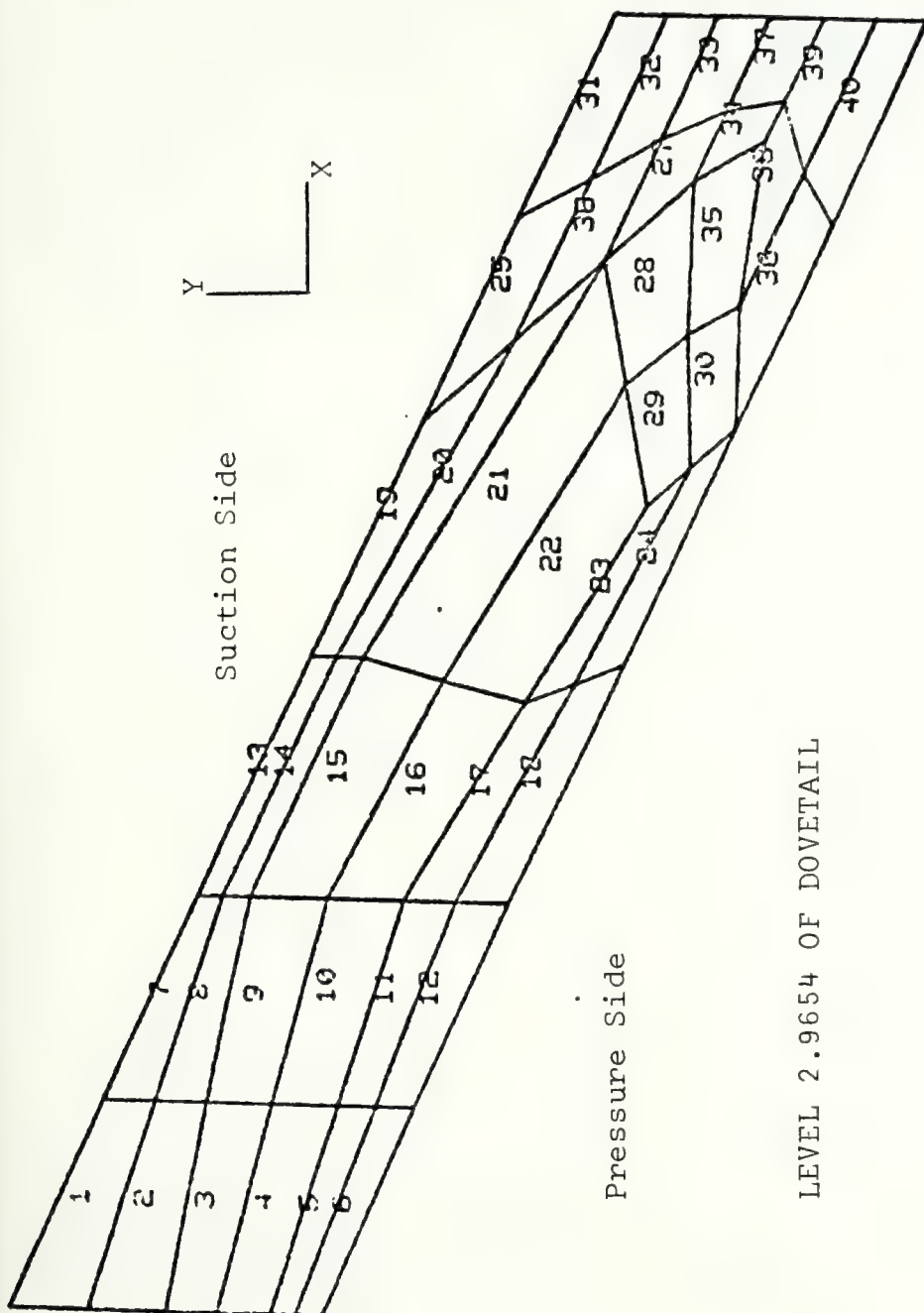
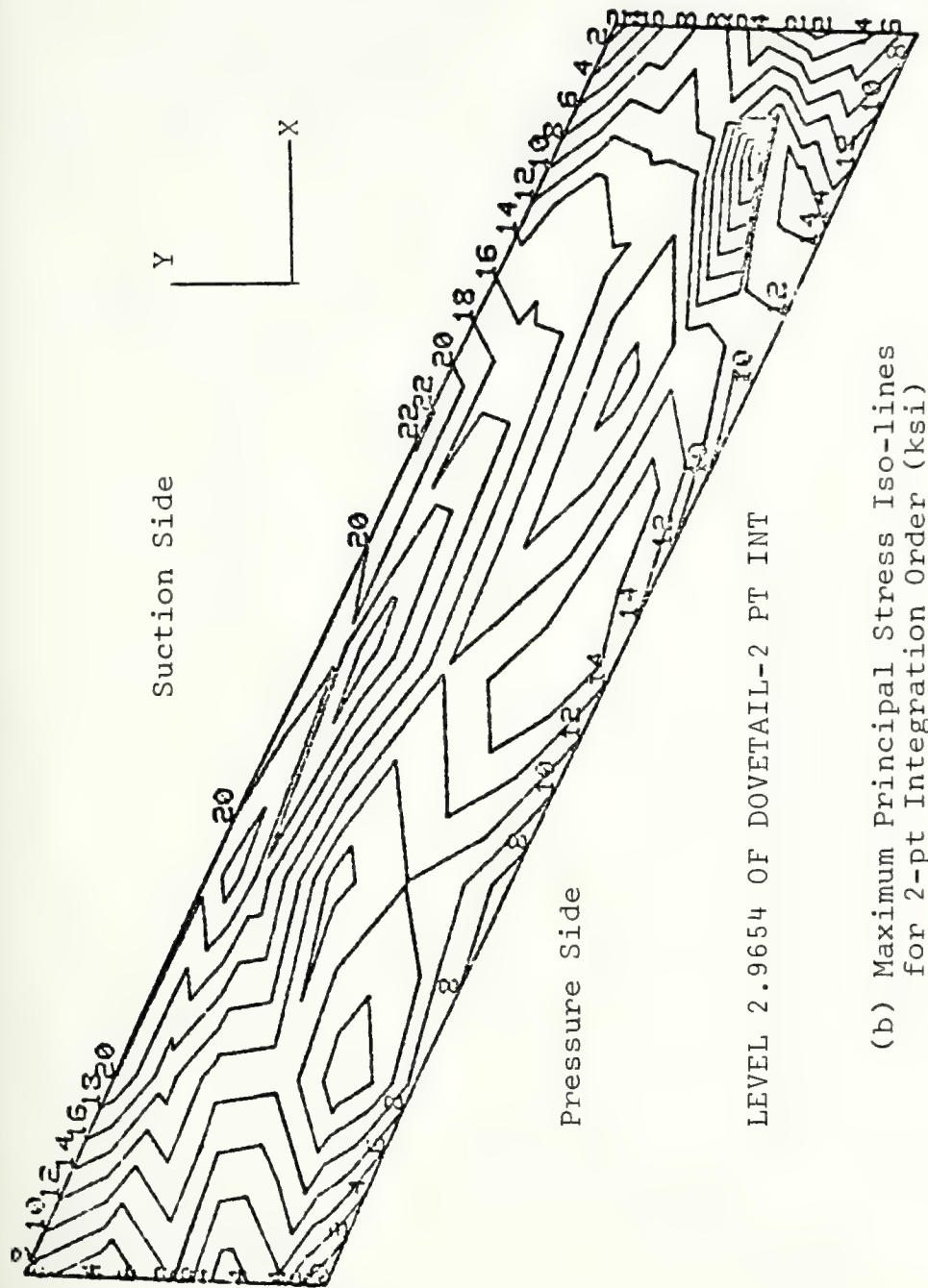


Figure 14. Continued, Page 3 of 3.



LEVEL 2.9654 OF DOVETAIL

(a) 2-D Element Arrangement
Figure 15. Level 2.9651, Page 1 of 3.

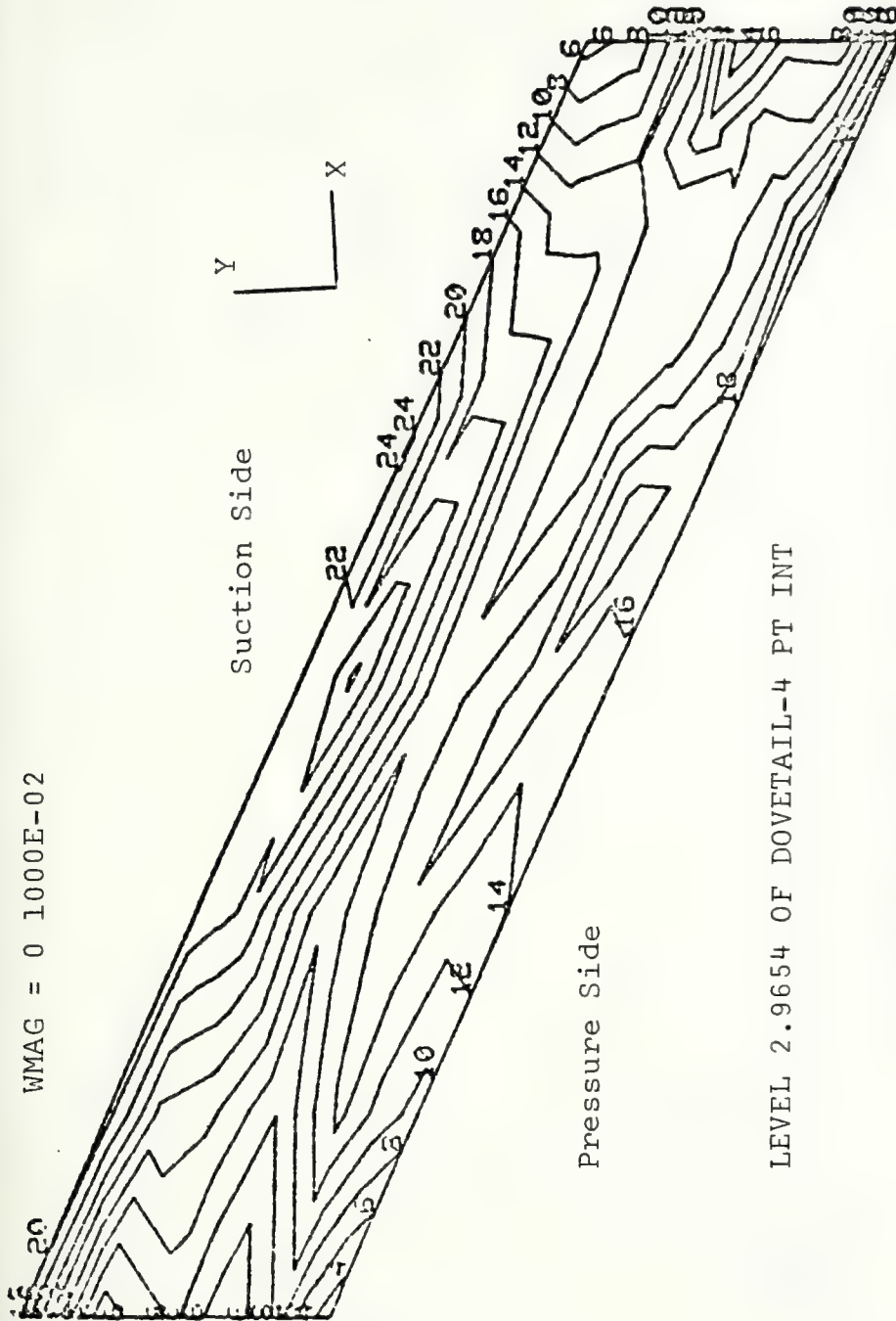


LEVEL 2.9654 OF DOVETAIL-2 PT INT

(b) Maximum Principal Stress Iso-lines
for 2-pt Integration Order (ksi)

Figure 15. Continued, Page 2 of 3.

WMAG = 0 1000E-02



Pressure Side

Suction Side

LEVEL 2.9654 OF DOVETAIL-4 PT INT

(c) Maximum Principal Stress Iso-lines
for 4-pt Integration Order (ksi)

Figure 15. Continued, Page 3 of 3.

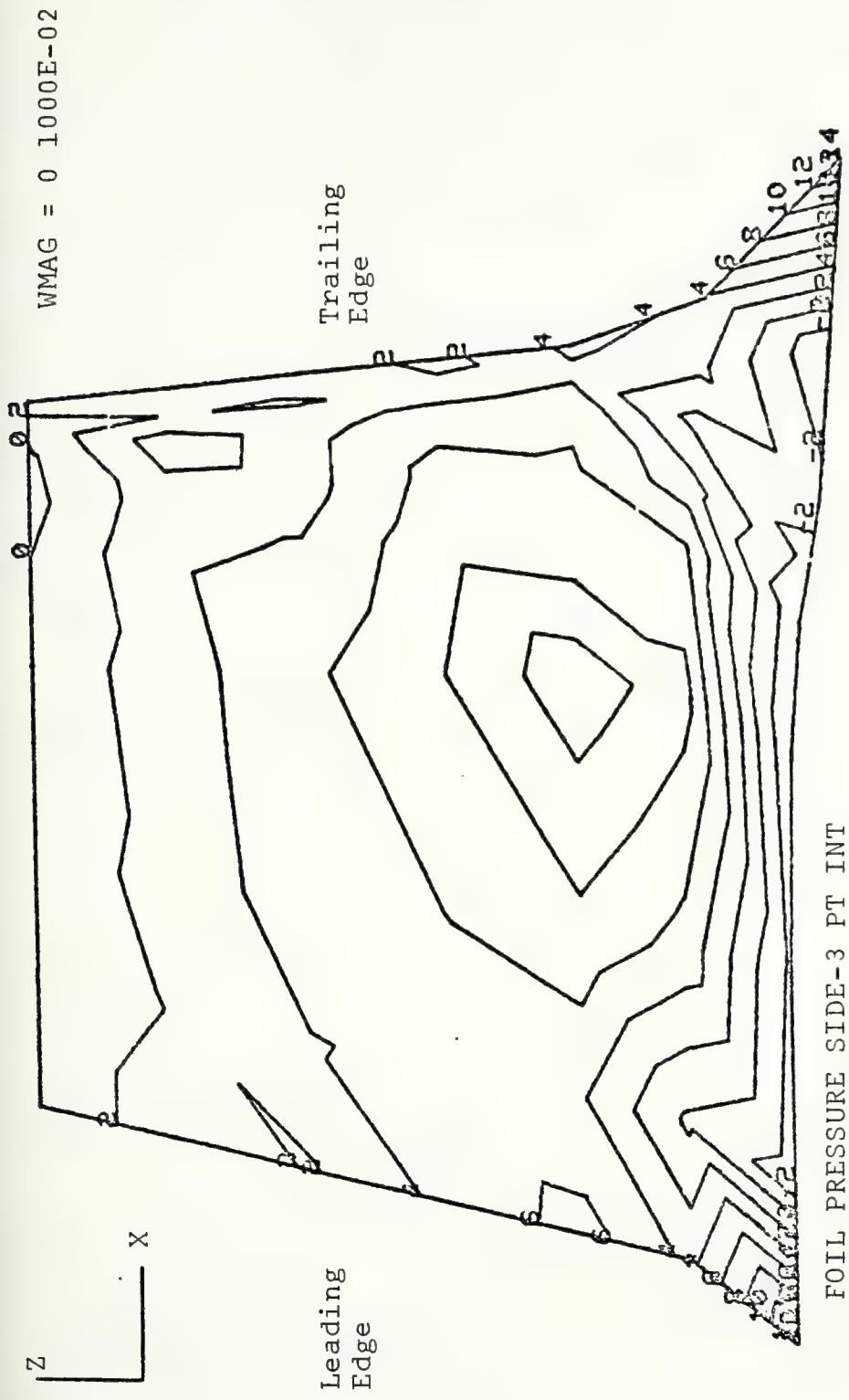


Figure 16. Maximum Principal Stress Contours for Pressure Side of Airfoil (ksi).

WMAG = 0 1000E-02

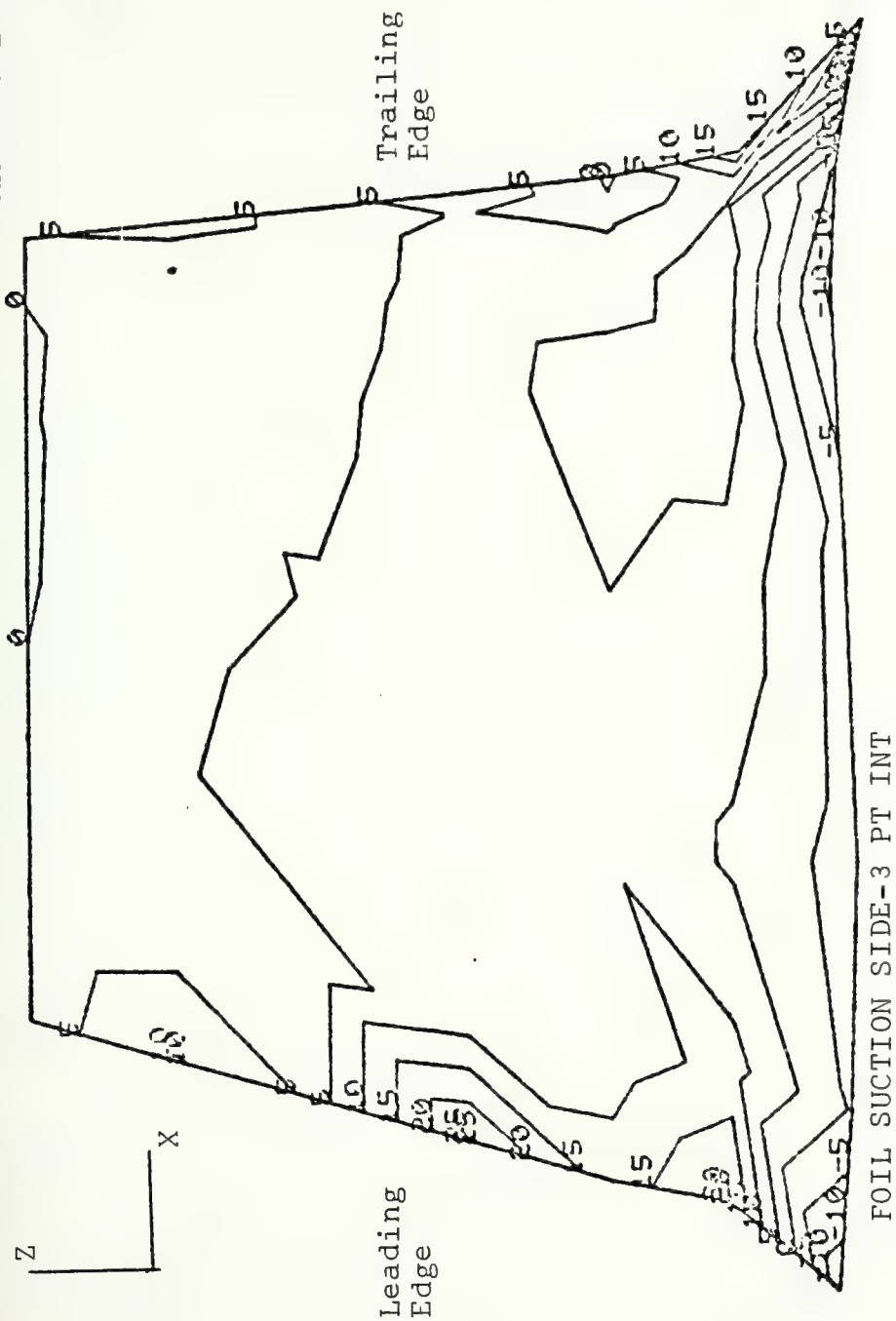


Figure 17. Maximum Principal Stress Contours for Suction Side of Airfoil (ksi).

VII. CONCLUSIONS

The time required for development of pre- and post-processors prevented a complete analysis which would have the potential for strong conclusive opinions and possible suggestions for design improvement. Software and hardware tools are, however, now implemented and identified which will allow follow-on analysis by NPS thesis students of geometric complex machine components. Despite the handicap of an inconclusive analysis, certain comments can be made concerning the results.

A. ANALYSIS RESULTS

Three other finite element analyses of ceramic turbine blade designs were reviewed by the author [Refs. 9, 10, 11]. Despite differing boundary conditions and loading schemes, a surface stress concentration persists in the region immediately above the blade-disk contact surface. This fact indicates that further research is required into the possibility of optimizing the dovetail geometry in order to spread the stresses over a greater bulk of material. If such an optimized design proves infeasible or non-existent, then the conclusion of this analysis and review indicates that hot-pressed Silicon-Nitride is unsuitable even for the limited goals of the current project. Using the four point bending strength of seventy-two ksi [Ref. 9] as a reference strength parameter, a simplistic factor of safety of 1.57 is realized. Considering the analysis did not impose

pressure and temperature gradient induced stresses, this factor of safety appears to be quite inadequate for predicting a decent probability of success.

Despite somewhat severe distortions in many of the analysis model elements, no mathematical singularities were encountered in using a reduced integration order. Further analysis using finer meshes may show that the reduced integration results are more accurate than the "exact" integration order. Should this be the case, approximately fifteen per cent higher values would be predicted than the results of "exact" integration of the stiffness matrix elements. These higher values would necessarily reduce the probability of success calculations normally used in design.

B. OPPORTUNITIES FOR FURTHER RESEARCH

The computer hardware and software at NPS, together with the groundwork laid by this thesis, allows for many avenues of follow-on research of which the following is a partial list:

- 1) refinement of the mesh of this analysis and convergence study of results,
- 2) analysis of the blade loaded under pressure and temperature gradient induced stresses,
- 3) optimization of attachment root design in order to alleviate stress concentrations,

- 4) probabilistic failure analysis using ADINA generated stresses,
- 5) frequency analysis of the blade,
- 6) investigation of appropriate boundary conditions,
- 7) analysis of the three material system of the blade-disk contact region.

APPENDIX A

ADDENDUM TO ADINA USER'S MANUAL, REPORT
82448-1, MIT, SEPTEMBER 1975
(Revised May 1976)

Two additional output options are available to the ADINA user at the Naval Postgraduate School besides what is described in the published user's manual. Master Control Card one, documented on page II.1, may be altered as follows:

NOTES	COLUMNS	VARIABLE	ENTRY
(a)	71-75	ITP57	Indicator for extended stress output and storage of stresses on user defined file 57. EQ.0; option not desired EQ.1; option desired
(b)	76-80	ITP58	Indicator for storage of displacements on user defined file 58. EQ.0; option not desired EQ.1; option desired

NOTES:

(a) The use of this option gives the user an output of stresses at all twenty-seven locations per element described in Section X.3 and illustrated on page X.37. Stresses are stored sequentially on user defined file 57 in the following order:

INPUT STEP

1. element number
2. integer 1 corresponding to figure X.5
3. normal stress for node in global X direction
4. normal stress for node in global Y direction
5. normal stress for node in global Z direction
6. shear stress XY
7. shear stress XZ
8. shear stress YZ

Steps 2 through 8 are repeated for the remainder of the twenty-seven nodes of the element, followed by a complete cycle for each of the remaining elements.

(b) Use of this option stores the six global displacements for each node on user defined file 58 in the following order:

INPUT STEP

1. node number
2. X direction displacement
3. Y direction displacement
4. Z direction displacement
5. X-axis rotation (if present)
6. Y-axis rotation (if present)
7. Z-axis rotation (if present)

Steps 1 through 7 are repeated for each of the input nodes. Displacements for the additional nodes for which stress results are calculated with option ITP57 are not calculated.

APPENDIX B

CONVERSION OF CALCOMP PLOTTING ROUTINES FOR USE ON A TEKTRONIX 4012 TERMINAL

The TEKTRONIX 4012 Computer Display Terminal is the most powerful interactive graphics system available to the general user at the Naval Postgraduate School. The major advantages of this unit are its speed and access to the IBM 360 computer via the CP/CMS system. A major disadvantage is there is only one terminal designated for the general user. During the final stages of preparation of this report, PSAP1 and Program Contour Plot were modified for use with the TEKTRONIX terminal in order to achieve a higher quality of plot in a timely manner than is possible with the CALCOMP system. These modifications proved quite simple and resulted in a usable plotting system. Presented in this appendix are the subroutines added to the graphics programs used by the author and a few general comments. The user should consult Refs. 12, 13, 14 and 15 for detailed information on using the TEKTRONIX unit.

The user must first determine the plotting origin of his routine and establish the limits of his plotting window using subroutine VWINDO. Caution must be used in order to ensure that the original program will properly scale the plotting values in both coordinate directions. CALCOMP PLOT statements which shift the origin must be deleted. Subroutines must then be added to the routine which are titled with CALCOMP plotting library routine names which in turn

call equivalent subroutines in the TEKTRONIX library. The following are examples used with PSAP1 and Program Contour Plot:

1) Subroutine equivalent to CALCOMP PLOTS:

```
SUBROUTINE PLOTS
  IBAUD=480
  CALL INITT (IBAUD)
  CALL BINITT
  RETURN
END
```

2) Subroutine equivalent to CALCOMP PLOTE:

```
SUBROUTINE PLOTE
  IX=0
  IY=780
  CALL FINITT (IX,IY)
  RETURN
END
```

3) Subroutine equivalent to CALCOMP LINE:

```
SUBROUTINE LINE (Y,X,N,N1,N2)
  DIMENSION X(21), Y(21)
  CALL MOVEA (X(1),Y(1))
  DO 10 I=2,N
    CALL DRAWA (X(I),Y(I))
  10 CONTINUE
  RETURN
END
```

4) Subroutine equivalent to CALCOMP NUMBER:

```
SUBROUTINE NUMBER (Y,X,H,AL,TH,N)
  DIMENSION IARRAY (4)
  CALL MOVEA (X,Y)
  CALL IFORM (AL,4,IARRAY,32)
  DO 10 I=1,4
    IF (IARRAY(I).EQ.32) GO TO 10
    CALL ANCHO (IARRAY(I))
  10 CONTINUE
  RETURN
END
```


5) Subroutine equivalent to CALCOMP SYMBOL:

```
SUBROUTINE SYMBOL (IY,IX,H,BCK,TH,N)
DIMENSION IARRAY (80), BCK (1)
CALL CONETA (BCK,N,IARRAY)
CALL NOTATE (IX,IY,N,IARRAY)
RETURN
END
```

6) Subroutine equivalent to CALCOMP PLOT:

```
SUBROUTINE PLOT (Y,X,I)
IF (I.EQ.2) CALL DRAWA (X,Y)
IF (I.EQ.3) CALL MOVEA (X,Y)
RETURN
END
```

These subroutines were tailored for use only with PSAP1 and Program Contour Plot and are much more restrictive than the CALCOMP equivalent. For example, subroutine NUMBER defined above will only plot integers up to four digits whereas subroutine NUMBER used with the CALCOMP will plot any integer or floating point number input by the user. The subroutines do, however, provide an example of the simple modifications required and the TEKTRONIX software is available for a much higher degree of graphics sophistication.

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APPENDIX C

PROGRAM CENTRIFUGAL LOAD LISTING

```

*****
PROGRAM CENTRIFUGAL LOAD
THIS PROGRAM READS AN ADINA INPUT DECK AND CALCULATES NODAL
CONSTANT LOADS CAUSED BY A GIVEN ROTATION ABOUT ANY COMBI-
NATION OF RIGHT-HAND CARTESIAN COORDINATE SYSTEM AXES.
VARIOUS CHOICES OF OUTPUTS CAN BE SELECTED INCLUDING A
PUNCHED CARD DECK IN A FORMAT ACCEPTABLE TO ADINA FINITE
ELEMENT CODE FOR INPUT OF CONCENTRATED NODAL FORCES.

INPUT DATA DECK
CARD 1 (FORMAT 4F10.0)
VARIABLE      COL      REMARKS
W1            1-10     ANGULAR VELOCITY ABOUT THE X-AXIS
W2            11-20    ANGULAR VELOCITY ABOUT THE Y-AXIS
W3            21-30    ANGULAR VELOCITY ABOUT THE Z-AXIS
RO            31-40    MASS DENSITY

CARD 2 (FORMAT 8I5)
NUMNF         1-5      NUMBER OF NODES
NUMEL         6-10     NUMBER OF ELEMENTS
NGP           11-15    ORDER OF GAUSS POINT INTEGRATION (2-6)
NCUR          16-20    LOAD CURVE NUMBER TO BE USED IN ADINA
ICHK1         21 25    IAW SECTION XII OF ADINA USERS MANUAL
                     FLAG FOR PRINTOUT OF CF NODE COORDINATE AND
                     MESH CONNECTIVITY DATA
                     0-NO PRINTOUT
                     1-PRINTOUT
ICHK2         26-30    FLAG FOR PRINTOUT OF CONSISTANT LOADS
                     BY ELEMENT
                     0-NO PRINTOUT
                     1-PRINTOUT
ICHK3         31-35    FLAG FOR PRINTOUT OF CF TOTALED CONSISTANT
                     LOADS FOR EACH NODE
                     0-NO PRINTOUT
*****

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CC


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CEN00410
CEN00420
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CEN00440
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CEN00460
CEN00470
CEN00480
CEN00490
CEN00500
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CEN00580
CEN00590
CEN00600
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CEN00670
CEN00680
CEN00690
CEN00700
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CEN00720
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CEN00780
CEN00790
CEN00800
CEN00820
CEN00830
CEN00840
CEN00850
CEN00860
CEN00870
CEN00880
CEN00890

1-PRINTOUT
FLAG FOR PUNCHED DECK OF NODE CONSISTANT
FORCES
0-NO PUNCHING
1-PUNCH LOAD DECK

PLACE ADINA INPUT DECK HERE. THE PORTION OF THE DECK REQUIRED
IS FROM THE TITLE CARD TO THE LAST CONNECTIVITY CARD INCLUSIVE.

A DYNAMIC DIMENSIONING SCHEME IS USED IN ORDER TO GENERALIZE THE
THE PROGRAM FOR VARIOUS MESHES. SHOULD MORE STORAGE SPACE
BE REQUIRED, THE FOLLOWING DIAGNOSTIC WOULD BE CUTPUT:
MTOT=XXXXXX INCREASE MTOT FOR ADDITIONAL STORAGE
SHOULD THIS DIAGNOSTIC OCCUR, CHANGE THE FOLLOWING TWO CARDS
IN THE MAIN PROGRAM:
COMMON A(XXXXXX)
MTOT=XXXXXX
INCREASE XXXXXX TO A VALUE GREATER THAN THAT GIVEN FOR MTOT
BY THE DIAGNOSTIC.

REFERENCES:
BATHE,KLAUS-JURGEN: ADINA(REPORT 82448-1):SEPT 1975,
(REVISION MAY 1976);MIT, CAMBRIDGE MASS.

EASTERLING,L.R.;FINITE ELEMENT ANALYSIS OF A CERAMIC GAS
TURBINE BLADE, THESIS REPORT; NPS,MONTEREY CA. MAR 1978

WRITTEN BY: PROF GILLES CANTIN AND L.R. EASTERLING,
NPS, MONTEREY CA. FEBRUARY 1978.

*****
IMPLICIT REAL*8 (A-H,O-Z)
*****
MAIN PROGRAM TO READ ADINA DECK AND CALCULATE CCNSISTANT
CENTRIFUCAL LOADS
*****
COMMON A(15000)
COMMON/ RPARA/W1, W2,W3,RO
COMMON/ IPARA/NUMNP,NUMEL,NGP,NCUF
COMMON/CHECK/ICHK1,ICHK2,ICHK3,ICHK4
MTOT=15000
READ(5,1000)W1,W2,W3,RO
FCRMAT(4F10.0)
1000

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1001 READ(5,1001)NUMNP,NUMEL,NGP,NCUR,ICLK1,ICLK2,ICLK3,ICLK4
      FCFORMAT(815)
      WRITE(6,1002)W1,W2,W3,RO
      WRITE(6,1003)NUMNP,NUMEL,NGP
1002 FCFORMAT(//5X,ANGULAR VELOCITIES//10X,WX = ',G25.16/10X,WY = ',
      1 G25.16/10X,WZ = ',G25.16//5X,DENSITY = ',G25.16//)
1003 FCFORMAT(//5X,NUMBER OF NODE PGINTS = ',15//5X,NUMBER OF
      2 ELEMENT = ',15//5X,NUMBER OF GAUSS POINTS FCR INTERGRATION
      3 = ',15//)
      N1=1
      N2=N1+60*NUMEL
      N3=N2+NUMNP
      N4=N3+NUMNP
      N5=N4+NUMNP
      N6=N5+NUMNP
      N7=N6+NUMEL
      N8=N7+20*NUMEL
      NLAST=N8+3*NUMNP
      IF(NLAST.GT.MTOT)WRITE(6,2000)NLA
2000 1,/,/,
      1 CALL CENTF(A(N1),A(N2),A(N3),A(N4),A(N5),A(N6),A(N7),A(N8))
      STOP
      END
      SUBROUTINE CENTF(TFVT,XPT,YPT,ZPT,NUMPT,NREL,NCONT,FORCE)
      *****
      SUBROUTINE TO COORDINATE INPUT AND OUTPUT CF
      CONSISTANT CENTRIFUGAL LOADS.
      *****
      IMPLICIT REAL*8 (A-H,O-Z)
      COMMON/ RPARA/W1,W2,W3,RO
      COMMON/ IPARA/NUMNP,NUMEL,NGP,NCUR
      COMMON/ LOADS/NLOAD,NLCUR,NPTM,IDGRAV
      COMMON/ CHECK/ICLK1,ICLK2,ICLK3,ICLK4
      DIMENSION XPT(1),YPT(1),ZPT(1),NUMPT(1),NREL(1)
      DIMENSION TFVT(20,3,1),FORCE(3,1),NCONT(20,1)
      CALL RDADIN(XPT,YPT,ZPT,NCONT,NREL,NUMPT)
      IF(ICLK1.EQ.0)GO TO 10
      CALL PNTOUT(XPT,YPT,ZPT,NCONT,NREL,NUMPT)
      60 CALL TRANS(TFVT,XPT,YPT,ZPT,NCONT,NREL,NUMPT,FORCE)
      10 RETURN
      END
      SUBROUTINE TRANS (TFVT,XPT,YPT,ZPT,NCONT,NREL,NUMPT,FORCE)
      CEN00900
      CEN00910
      CEN00920
      CEN00930
      CEN00940
      CEN00950
      CEN00960
      CEN00970
      CEN00980
      CEN00990
      CEN01000
      CEN01010
      CEN01020
      CEN01030
      CEN01040
      CEN01050
      CEN01060
      CEN01070
      CEN01080
      CEN01090
      CEN01100
      CEN01110
      CEN01120
      CEN01130
      CEN01140
      CEN01150
      CEN01160
      CEN01170
      CEN01180
      CEN01190
      CEN01200
      CEN01210
      CEN01220
      CEN01230
      CEN01240
      CEN01250
      CEN01260
      CEN01270
      CEN01280
      CEN01290
      CEN01300
      CEN01310
      CEN01320
      CEN01330
      CEN01340
      CEN01350
      CEN01360
      CEN01370

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CEN018670  
CEN01879  
CEN01880  
CEN01890  
CEN01900  
CEN01910  
CEN01920  
CEN01930  
CEN01940  
CEN01950  
CEN01960  
CEN01970  
CEN01980  
CEN01990  
CEN02000  
CEN02010  
CEN02020  
CEN02030  
CEN02040  
CEN02050  
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CEN02070  
CEN02080  
CEN02090  
CEN02100  
CEN02110  
CEN02120  
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CEN02160  
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CEN02180  
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CEN02220  
CEN02230  
CEN02240  
CEN02250  
CEN02260  
CEN02270  
CEN02280  
CEN02290  
CEN02300  
CEN02310  
CEN02320  
CEN02330
```

```
XTCOT2=0.0D0  
YTCOT2=0.0D0  
ZTCOT2=0.0D0  
DC 555 I=1,NUMNP E(1,I)  
XTCOT2=XTCOT2+FORCE(1,I)  
YTCOT2=YTCOT2+FORCE(2,I)  
ZTCOT2=ZTCOT2+FORCE(3,I)  
IF(ICCHK3.EQ.O)GO TO 888  
DC 666 I=1,NUMNP  
WRITE(6,1006)I,(FORCE(J,I),J=1,3)  
FCRMA T(/ /4X,'NODE ',I5,2X,'FX = ',G25.16,2X,  
    1,FZ = ',G25.16)  
CONTINUE  
666 WRITE(6,222)XTOT2,YTOT2,ZTOT2  
888 KLCAD=O  
IF(ICHK 4.EQ.O)GO TO 999  
DC 777 I=1,NUMNP  
DC 777 J=1,3  
IF(DABS(FORCE(J,I)).LT.O.1D-8)GO TO 777  
KLCAD=KLOAD+1  
WRITE(7,1004)I,J,NCUR,FORCE(J,I)  
CONTINUE  
777 WRITE(6,1003)KLOAD  
FCRMA T(/ /5X,'CONSISTANT CENTRIFUGAL LOADS FOR ELEMENT ',I5//)  
1000 FCRMA T(5X,'NATURAL',5X,'GLOBAL',5X,'COORD NR',4X,'CCORD NR',7X,  
    1,X-COMPONENT',14X,'Y-COMPONENT',14X,'Z-COMPONENT'//)  
1002 FCRMA T(O,6X,I5,6X,I5,3G30.16)  
1003 FCRMA T(/ /5X,'TOTAL ADDITIONAL NLCAD REQUIRED IS ',I10//)  
1004 FCRMA T(3I5,G25.16)  
10C5 FCRMA T(4I5)  
RETURN  
END  
SUBROUTINE CUBAT(COD,FVT,W1,W2,W3,RO,NCON,NGF,IEL)  
  
      CUBAT FORMS THE CONSISTENT LOAC VECTOR CORRESPONDING TO THE  
      ANGULAR VELOCITY CF COMPONENT W1,W2,W3 ABOUT THE ORIGIN  
      PROF GILLES CANTIN AT THE NAVAL POSTGRDUATE SCHOOL SEPT. 1977  
  
      IMPLICIT REAL*(A-H,O-Z)  
      DIMENSION COD(20,3),FVT(20,3),DUM(20,3),GP(6,5),WF(6,5),NCON(20)  
      DATA GP/,57735026918963DO,-.57735026918963DO,0.3DO,0.3DO,0.3DO,0.  
    10,7745966924148DO,C.O DO,-.7745966924148DO,0.3DO,0.3DO,0.3DO,0.  
    211,3631159405DO,0.33998104358486DO,-.33998104358486DO,-.861136311599C  
    3405DO,0.ODO,0.ODO,,90617984593866DO,.53846931010568DO,0.ODO,-.5384C  
    46531010568DO,-.9061758459386DO,0.ODO,.93246951420315DO,.66120938C  
    5646626DO,2.23861918608317DO,-.23861918608317DO,-.06120938646626DO,-  
    6.53246951420315DO/  
      DATA WF/,1.ODO,1.ODO,0.ODO,0.ODO,0.ODO,0.ODO,.555555555555556DO,.388C
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CEN02820
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CEN02970
CEN02980
CEN02990
CEN03000
CEN03010
CEN03020
CEN03030
CEN03040
CEN03050
CEN03060
CEN03070
CEN03080
CEN03090
CEN03100
CEN03110
CEN03120
CEN03130
CEN03140
CEN03150
CEN03160
CEN03170
CEN03180
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CEN03200
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CEN03220
CEN03230
CEN03240
CEN03250
CEN03260
CEN03270
CEN03280
CEN03290

COD IS AN INPUT ARRAY OF THE ELEMENT COORDINATES IN THE
GLOBAL COORDINATE SYSTEM
W1 IS THE COMPONENT AF ANGULAR VELOCITY ABOUT THE AXIS X
W2 IS THE COMPONENT AF ANGULAR VELOCITY ABOUT THE AXIS Y
W3 IS THE COMPONENT AF ANGULAR VELOCITY ABOUT THE AXIS Z
NCON IS THE CONNECTIVITY MATRIX OF THE ELEMENT
IEL IS THE ELEMENT IDENTIFICATION NUMBER

PROF. GILLES CANTIN SEPT. 1977 NAVAL POSTGRADUATE SCHOOL
MONTEREY CALIFORNIA 93940

IMPLICIT REAL*8(A-H,O-Z)
DIMENSION COD(20,3),DNI(3,20),
DIMENSION FVT(20,3),A(3,3),NCON(20)
REAL*8 JAC(3,3)
GM(BT) = 0.500*(1.000 + BT)
GZ(BT) = 0.500*(1.000 - BT)
GZ(BT) = 1.000 - BT*BT
DP(BT) = 0.500 + ZRC*BT
DM(BT) = -0.500 + ZRC*BT
DZ(BT) = -2.000*BT
ZRO = 0.000
ONE = 1.000
TWO = 2.000
DO 100 I=1,3
DO 100 J=1,20
FVT(I,J) = ZRO
DNI(I,J) = GZ(R)*GP(S)*GP(T) = ZRO
IF(NCON(I,9).EQ.0) EMI(9) = ZRO
EMI(10) = GM(R)*GZ(S)*GP(T) = ZRO
IF(NCON(I,10).EQ.0) EMI(10) = ZRO
EMI(11) = GZ(R)*GM(S)*GP(T) = ZRO
IF(NCON(I,11).EQ.0) EMI(11) = ZRO
EMI(12) = GP(R)*GZ(S)*GP(T) = ZRO
IF(NCON(I,12).EQ.0) EMI(12) = ZRO
EMI(13) = GZ(R)*GP(S)*GM(T) = ZRO
IF(NCON(I,13).EQ.0) EMI(13) = ZRO
EMI(14) = GM(R)*GZ(S)*GM(T) = ZRO
IF(NCON(I,14).EQ.0) EMI(14) = ZRO
EMI(15) = GZ(R)*GM(S)*GM(T) = ZRO
IF(NCON(I,15).EQ.0) EMI(15) = ZRO
EMI(16) = GP(R)*GZ(S)*GM(T) = ZRO
IF(NCON(I,16).EQ.0) EMI(16) = ZRO
EMI(17) = GP(R)*GP(S)*GZ(T) = ZRO
IF(NCON(I,17).EQ.0) EMI(17) = ZRO
EMI(18) = GM(R)*GP(S)*GZ(T) = ZRO
IF(NCON(I,18).EQ.0) EMI(18) = ZRO

CCCCCCCC

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 CEN033310
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 CEN033360
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 CEN033710
 CEN033720
 CEN033730
 CEN033740
 CEN033750
 CEN033760
 CEN033770

ENI(19) = GM(R)*GM(S)*GZ(T)
 IF(NCON(19).EQ.0) ENI(19) = ZRO
 IF(NCON(20).EQ.0) ENI(20) = ZRO
 ENI(1) = GP(R)*GP(S)*GP(T)
 ENI(2) = GM(R)*GM(S)*GP(T)
 ENI(3) = GM(R)*GM(S)*GP(T)
 ENI(4) = GM(R)*GM(S)*GP(T)
 ENI(5) = GM(R)*GM(S)*GM(T)
 ENI(6) = GM(R)*GM(S)*GM(T)
 ENI(7) = GM(R)*GM(S)*GM(T)
 ENI(8) = GM(R)*GM(S)*GM(T)
 IF(NCON(9).EQ.0) GO TO 500
 DNI(1,9) = DZ(R)*GP(S)*GP(T)
 DNI(2,9) = GZ(R)*GP(S)*GP(T)
 DNI(3,9) = GZ(R)*GP(S)*DP(T)
 IF(NCON(10).EQ.0) GO TO 510
 DNI(1,10) = DM(R)*GZ(S)*GP(T)
 DNI(2,10) = GM(R)*DZ(S)*GP(T)
 DNI(3,10) = GM(R)*GZ(S)*DP(T)
 IF(NCON(11).EQ.0) GO TO 520
 DNI(1,11) = DZ(R)*GM(S)*GP(T)
 DNI(2,11) = GZ(R)*DM(S)*GP(T)
 DNI(3,11) = GZ(R)*GM(S)*DP(T)
 IF(NCON(12).EQ.0) GO TO 530
 DNI(1,12) = DP(R)*GZ(S)*GP(T)
 DNI(2,12) = GP(R)*DZ(S)*GP(T)
 DNI(3,12) = GP(R)*GZ(S)*DP(T)
 IF(NCON(13).EQ.0) GO TO 540
 DNI(1,13) = DZ(R)*GP(S)*GM(T)
 DNI(2,13) = GZ(R)*DP(S)*GM(T)
 DNI(3,13) = GZ(R)*GP(S)*DM(T)
 IF(NCON(14).EQ.0) GO TO 550
 DNI(1,14) = DM(R)*GZ(S)*GM(T)
 DNI(2,14) = GM(R)*DZ(S)*GM(T)
 DNI(3,14) = GM(R)*GZ(S)*DM(T)
 IF(NCON(15).EQ.0) GO TO 560
 DNI(1,15) = DZ(R)*GM(S)*GM(T)
 DNI(2,15) = GZ(R)*DM(S)*GM(T)
 DNI(3,15) = GZ(R)*GM(S)*DM(T)
 IF(NCON(16).EQ.0) GO TO 570
 DNI(1,16) = DP(R)*GZ(S)*GM(T)
 DNI(2,16) = GP(R)*DZ(S)*GM(T)
 DNI(3,16) = GP(R)*GZ(S)*DM(T)
 IF(NCON(17).EQ.0) GO TO 580
 DNI(1,17) = DP(R)*GP(S)*GZ(T)
 DNI(2,17) = GP(R)*DP(S)*GZ(T)
 DNI(3,17) = GP(R)*GP(S)*DZ(T)

+ + + + +
 EMI(9)
 EMI(10)
 EMI(11)
 EMI(13)
 EMI(14)
 EMI(15)
 EMI(16)
 I(12)
 I(10)
 I(11)
 EMI(12)
 EMI(16)
 EMI(14)
 EMI(15)
 I(17))//TWO
 I(18))//TWO
 I(19))//TWO
 I(20))//TWO
 EMI(17))//TWO
 EMI(18))//TWO
 EMI(19))//TWO
 EMI(20))//TWO

500
 510
 520
 530
 540
 550
 560
 570


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CEN04740
CEN04750
CEN04760
CEN04770
CEN04780
CEN04790
CEN04800
CEN04810
CEN04820
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CEN04890
CEN04900
CEN04910
CEN04920
CEN04930
CEN04940
CEN04950
CEN04960
CEN04970
CEN04980
CEN04990
CEN05000
CEN05010
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CEN05100
CEN05110
CEN05120
CEN05130
CEN05140
CEN05150
CEN05160
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CEN05180
CEN05190
CEN05200
CEN05210

WRITE(6,17)
17 FCRMAT(5X,'RESEQUENCED',4X,'USER INPUT',
15X,'GRID POINT',5X,'GRID POINT',/
25X,'NUMBER',9X,'NUMBER',13X,'X',14X,'Y',14X,'Z',//)
DO 30 I=1,NUMNP
18 WRITE(6,18) I,NUMPT(1),XPT(1),YPT(1),ZPT(1)
30 FCRMAT(2X,I10,5X,I10,3X,3E15.4)
30 CONTINUE
19 WRITE(6,19)
19 FCRMAT(///,5X,'ELEMENT INFORMATION - WITH RESEQUENCED GRID PCINTS
1,///)
9008 WRITE(6,9008)
FCRMAT(1X,'RESEQUENCED',4X,'USER INPUT',25X,'GRID POINTS',/
11X,'ELEMENT',8X,'ELEMENT',/
21X,'NUMBER',9X,'NUMBER',7X,'
3 8 9 10 11 12 13 14 15 16 17 18 19 20'//)
DC 40 I=1,NUMEL
WRITE(6,9010)I,NREL(1),(NCONT(J,I),J=1,20)
9010 FCRMAT(1X,I4,11X,I4,9X,20I5)
40 CONTINUE
959 RETURN
END
SUBROUTINE READIN(XPT,YPT,ZPT,NCCNT,NREL,NUMPT)
*****
***** SUBROUTINE TO READ ADINA DECK.
*****
*****
IMPLICIT REAL*8 (A-H,O-Z)
REAL*4 TITLE(20)
COMMON/ IPARA/NUMNP,NUMEL,NGP,NCUR
COMMON/ LOADS/NLOAD,NLCUR,NPTM,ICGRAV
DIMENSION XPT(1),YPT(1),ZPT(1),NUMPT(1),NREL(1),NCONT(20,1)
DIMENSION IDOF(6),ID(6),IDOLD(6),NODE(20),NPAR(20)
1 NPAR(20)=INP(20)
DATA CTEST/,X
NCARD=0
READ(5,9000)(TITLE(I),I=1,20)
WRITE(6,9011)(TITLE(I),I=1,20)
FCRMAT(///,5X,20A4,///)
9011 READ MASTER CONTROL CARDS
C **** NUMNP = TOTAL NUMBER OF NODE POINTS
C **** NLTYP = TOTAL NUMBER OF ELEMENT GROUPS
C **** READ(5,9001) NUMNP,(IDOF(I),I=1,6),NEGL,NEGAL,MODEX,NSTE
9001 FCRMAT(15,6I1,14,3I5)
NLTYP=NEGL+NEGAL

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K=K+KNOLD
XPT(K)=XPT(KK)+CX
IF(CT.NE.CTEST) GO TO 26
ROLD=ROLD+DR
DUMOLD=DUMOLD+DT
YPT(K)=ROLD*DCOS(DUMOLD)
ZPT(K)=ROLD*DSIN(DUMOLD)
GO TO 28
CONTINUE
26 YPT(K)=YPT(KK)+DY
ZPT(K)=ZPT(KK)+DZ
CONTINUE
28 NUMPT(K)=K
CONTINUE
30 NCLD=N
KNOLD=KN
DUMOLD=DUM
TC COUNT DOFS TO DETERMINE NUMBER CF IC CARDS
C *** DC 55 I=1,6
IF(IDOF(I).EQ.C.AND.ID(I).EQ.0) NEQ=NEQ+1
IDOLD(I)=ID(I)
CONTINUE
55 IF(N.NE.NUMPT) GO TO 10
READ LOAD CCNTROL CARDS
READ(5,9012)NLOAD,NLCUR,NPTM,IDGRAV
FCRMAT(415)
9012 DC 80 I=1,IMASSN
IF(IMASSN.EQ.0) GO TO 81
READ(5,9000) DUMMY
CONTINUE
80 CONTINUE
81 IF(IDAMPN.EQ.0) GO TO 91
DC 90 I=1,ICAMPN
READ(5,9000) DUMMY
CONTINUE
90 CONTINUE
91 READ INITIAL CONDITIONS
READ(5,9002) ICON
IF(ICON.EQ.C) GO TO 100
CARDNR=NEQ/6.0
NCARD=IDINT(CARDNR)
TEST=CARDNR-NCARD
IF(TEST.GT.C.1) NCARD=NCARD+1
DC 95 I=1,NCARD
READ(5,9000) DUMMY
CONTINUE
95 IF(IMASS.EQ.0) GO TO 100
DC 96 I=1,NCARD

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CEN05700
CEN05710
CEN05720
CEN05730
CEN05740
CEN05750
CEN05760
CEN05770
CEN05780
CEN05790
CEN05800
CEN05810
CEN05820
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CEN05990
CEN06000
CEN06010
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CEN06070
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CEN06090
CEN06100
CEN06110
CEN06120
CEN06130
CEN06140
CEN06150
CEN06160
CEN06170


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56      READ(5,9000) DUMMY
        CONTINUE
        DO 98 I=1,NCARD
          READ(5,9000) DUMMY
          CONTINUE
6007      FCRMAT(6E12.6)
100      CONTINUE
        NUMEL=0
9009      WRITE(6,9005) NEQ,NCARD
C ***      FCRMAT(//,,' NEQ AND NCARD FOR IC IN GEOM1 = ',15,10X,15//)
          READ ELEMENT CONTRCL CARDS
          DO 900 M=1,NELTYP
            REAC(5,9008)(NPAR(I),I=1,20)
            WRITE(6,9010) (NPAR(I),I=1,20)
9008      FCRMAT(20I4)
9010      FORMAT(//,,' NPAR = ',20I5//)
          MTYPE=NPAR(1)
          NUMMAT=NPAR(16)
          NSTRES=NPAR(13)
C ***      CALCULATE THE NUMBER OF MATERIAL CASE CARDS
          IF(NPAR(15).EQ.1) NCARD=1
          IF(NPAR(15).EQ.2) NCARD=2+NPAR(18)
          IF(NPAR(15).EQ.3) NCARD=4
          IF(NPAR(15).EQ.4) NCARD=4
          IF(NPAR(15).EQ.5) NCARD=2
          IF(NPAR(15).EQ.8) NCARD=1
          IF(NPAR(15).EQ.9) NCARD=1
          IF(NPAR(15).EQ.10) NCARD=6
          IF(NPAR(15).EQ.11) NCARD=6
          IF(NPAR(15).NE.12) GO TO 111
          CARDNR=NPAR(17)/8.0
          NCARD=IDINT(CARDNR)
          TEST=CARDNR-NCARD
          IF(TEST.GT.C.1) NCARD=NCARD+1
111      CONTINUE
C ***      N20=20 MATERIAL PROPERTIES
          DO 222 J=1,NUMMAT
            READ(5,9000) DUMMY
9000      FCRMAT(20A4)
            DO 45 I=1,NCARD
              READ(5,9000) DUMMY
              CONTINUE
45      CONTINUE
222      READ STRESS OUTPUT TABLE CARDS
C ***      IF(NPAR(13).EQ.0) GO TO 61
          DO 60 I=1,NSTRES
            READ(5,9000) DUMMY

```

CEN06180
 CEN06190
 CEN06200
 CEN06210
 CEN06220
 CEN06230
 CEN06240
 CEN06250
 CEN06260
 CEN06270
 CEN06280
 CEN06290
 CEN06300
 CEN06310
 CEN06320
 CEN06330
 CEN06340
 CEN06350
 CEN06360
 CEN06370
 CEN06380
 CEN06390
 CEN06400
 CEN06410
 CEN06420
 CEN06430
 CEN06440
 CEN06450
 CEN06460
 CEN06470
 CEN06480
 CEN06490
 CEN06500
 CEN06510
 CEN06520
 CEN06530
 CEN06540
 CEN06550
 CEN06560
 CEN06570
 CEN06580
 CEN06590
 CEN06600
 CEN06610
 CEN06620
 CEN06630
 CEN06640
 CEN06650

CEN06660
CEN06670
CEN06680
CEN06690
CEN06700
CEN06710
CEN06720
CEN06730
CEN06740
CEN06750
CEN06760
CEN06770
CEN06780
CEN06790
CEN06800
CEN06810
CEN06820
CEN06830
CEN06840
CEN06850
CEN06860
CEN06870
CEN06880
CEN06890
CEN06900
CEN06910
CEN06920
CEN06930
CEN06940
CEN06950
CEN06960
CEN06970
CEN06980
CEN06990
CEN07000
CEN07010

```

6C      CONTINUE
61      CCNTINUE
        IF(NPAR(14).EQ.0) NPAR(14)=1
        NEL=NPAR(14)-1
130     READ(5,5002) INEL,IINC
9002     FCRMAT(15,30X,I5)
        IF(IINC.EQ.C) IINC=1
9004     READ(5,9004) (INP(I),I=1,8)
14C     READ(5,9004) (INP(I),I=9,N20)
        FCRMAT(12I5)
        NEL=NEL+1
        NREL(NEL)=NEL
        ML=INEL-NEL
150     IF(ML) 150,155,160
800C    WRITE(6,800C)NEL
C *** NO GENERATION OF NCDE POINTS REQUIRED
155     CC 156 I=1,N20
        NP(I)=INP(I)
        NCONT(I,NEL)=NP(I)
        CONTINUE
156     GC TO 162
C *** GENERATION CF NODE PCINTS REQUIRED
160     DC 161 I=1,N20
        IF(NP(I).EQ.0) GO TO 161
        NP(I)=NP(I)+KN
        NCONT(I,NEL)=NP(I)
        CONTINUE
161     CCNTINUE
162     NLNREL=NLNREL+1
        IF(NEL.EQ.NPAR(2)) RETURN
        IF(NEL.LT.INEL) GO TO 140
        KN=IINC
        GO TO 130
900     CONTINUE
        END

```


PROGRAM KCONT LISTING

```

*****
PROGRAM KCONT
THIS PROGRAM READS AN 8-20 NODE ADINA INPUT DECK AND
DETERMINES THE UNIDENTIFIED NCGE LOCATIONS TO YIELD
THE COORDINATES AND CONNECTIVITY FOR A 27 NODE BRICK
MESH. THE OUTPUT IS PRINTED ON THE LINE PRINTER
AND WRITTEN ON USER DEFINED FILES 58 AND 59.

FILES:
FT58FC01 - COORDINATES OF ALL NODES WRITTEN IN
SEQUENCE AS FOLLOWS: NCGE NUMBER,
X-COORDINATE, Y-COORDINATE, Z-COORDINATE.

FT59FC01 - CONNECTIVITY ARRAY DIMENSIONED 27 BY
NUMEL. STORED IN SEQUENCE BY ROWS.

INPUT DATA REQUIREMENTS:

CARD 1
VARIABLE COLUMN REMARKS
NUMNP 1-10 NUMBER OF NODES IN INPUT
MESH
NUMEL 11-20 NUMBER OF ELEMENTS IN INPUT
MESH
PLACE HERE THE ADINA INPUT DECK FROM THE TITLE CARD
TO THE LAST CONNECTIVITY CARD INCLUSIVE.

WRITTEN BY: L.R. EASTERLING NPS, MONTEREY CA., FEB 1978

*****
COMMON A(30C00)
DATA A/3000C#0/
COMMON/ IPARA/NUMNP, NUMEL
READ(5,2000) NUMNP, NUMEL
FCRMAT(2110)
MTOT=30000
N1=1
2000

```

```

EAS00010
EAS00020
EAS00030
EAS00040
EAS00050
EAS00060
EAS00070
EAS00080
EAS00090
EAS00100
EAS00110
EAS00120
EAS00130
EAS00140
EAS00150
EAS00160
EAS00170
EAS00180
EAS00190
EAS00200
EAS00210
EAS00220
EAS00230
EAS00240
EAS00250
EAS00260
EAS00270
EAS00280
EAS00290
EAS00300
EAS00310
EAS00320
EAS00330
EAS00340
EAS00350
EAS00360
EAS00370
EAS00380
EAS00390
EAS00400
EAS00410

```


[illegible]

S009500
 EAS009100
 EAS009200
 EAS009300
 EAS009400
 EAS009500
 EAS009600
 EAS009700
 EAS009800
 EAS009900
 EAS010000
 EAS010100
 EAS010200
 EAS010300
 EAS010400
 EAS010500
 EAS010600
 EAS010700
 EAS010800
 EAS010900
 EAS011000
 EAS011100
 EAS011200
 EAS011300
 EAS011400
 EAS011500
 EAS011600
 EAS011700
 EAS011800
 EAS011900
 EAS012000
 EAS012100
 EAS012200
 EAS012300
 EAS012400
 EAS012500
 EAS012600
 EAS012700
 EAS012800
 EAS012900
 EAS013000
 EAS013100
 EAS013200
 EAS013300
 EAS013400
 EAS013500
 EAS013600
 EAS013700

```

NN2=NUMAP
DC 30 I=1, NUMEL
NN2=NN2+1
DC 40 J=1, 20
NCCN(J)=NCONT(J,I)
K=NCON(J)
CCD(J,1)=0.000
CCD(J,2)=0.000
CCD(J,3)=0.000
IF(K.EQ.0)GC TO 40
CCD(J,1)=XPT(K)
CCD(J,2)=YPT(K)
CCD(J,3)=ZPT(K)
CONTINUE
40 CALL ADCRD(R,S,T,COD,NCON,I,XX,YY,ZZ)
XFT(NN2)=XX
YFT(NN2)=YY
ZPT(NN2)=ZZ
30 KCONT(21,I)=NN2
DC 50 I=1, NUMEL
NFACE(J,I)=0.000
DC 50 K=1, 4
L=IFACE(K,J)
NFACE(J,I)=NFACE(J,I)+KCONT(L,I)
50 DC 3000 I=1, NUMEL
DC 3000 J=1, 12
I1=ILINE(1,J)
I2=ILINE(2,J)
NLINE(J,I)=KCONT(I1,I)+KCONT(I2,I)
3000 DC 70 I=1, NUMEL
DC 70 J=1, 15
K=NORD(J)
IF(KCONT(K,I).NE.0) GO TO 70
NA2=NN2+1
R=CNAT(1,J)
S=CNAT(2,J)
T=CNAT(3,J)
DC 80 M=1, 20
CCC(M,1)=0.000
CCC(M,2)=0.000
CCC(M,3)=0.000
NCCN(M)=NCONT(M,I)
N=NCON(M)
IF(N.EQ.0)GC TO 80
80 CCD(M,1)=XPT(N)
CCD(M,2)=YPT(N)
CCD(M,3)=ZPT(N)

```


EAS01380
 EAS01390
 EAS01400
 EAS01410
 EAS01420
 EAS01430
 EAS01440
 EAS01450
 EAS01460
 EAS01470
 EAS01480
 EAS01490
 EAS01500
 EAS01510
 EAS01520
 EAS01530
 EAS01540
 EAS01550
 EAS01560
 EAS01570
 EAS01580
 EAS01590
 EAS01600
 EAS01610
 EAS01620
 EAS01630
 EAS01640
 EAS01650
 EAS01660
 EAS01670
 EAS01680
 EAS01690
 EAS01700
 EAS01710
 EAS01720
 EAS01730
 EAS01740
 EAS01750
 EAS01760
 EAS01770
 EAS01780
 EAS01790
 EAS01800
 EAS01810
 EAS01820
 EAS01830
 EAS01840
 EAS01850

```

8C CCNTINUE
  CALL ADCRD(R,S,T,COD,NCON,I,XX,YY,ZZ)
  XPT(NN2)=XX
  YPT(NN2)=YY
  ZPT(NN2)=ZZ
  IF(K.LE.20)GO TO 5000
  DC 90 I I=1,NUMEL
  DC 90 J J=1,6
  KK=NORD(JJ+13)
  IF(NFACE(JJ,II).NE.NFACE(J-13,I))GC TO 90
  KTR=0
  DC 111 L=1,4
  LL=IFACE(L,J-13)
  DC 111 MM=1,8
  IF(KCONT(MM,II).EQ.KCONT(LL,I))KTR=KTR+1
  CCNTINUE
111 IF(KTR.EQ.4)KCONT(KK,II)=NN2
  50 CCNTINUE
  GC TO 70
5000 CCNTINUE
  DC 4000 I I=1,NUMEL
  DC 4000 J5=1,12
  KK=NORD(J5)
  IF(NLINE(J5,II).NE.NLINE(J,I))GC TC 4000
  KTR=0
  DC 4001 L=1,2
  LL=ILINE(L,J)
  DC 4001 MM=1,8
  IF(KCONT(MM,II).EQ.KCONT(LL,I))KTR=KTR+1
  CCNTINUE
4001 IF(KTR.EQ.2)KCONT(KK,II)=NN2
  CCNTINUE
4000 CCNTINUE
  70 WRITE(6,1003)NN2
  FCRMAT(//5X,I7,' DISTINCT POINTS ARE IDENTIFIED'//)
  C
  C
  C
1003 REWIND 58
  REWIND 59
  WRITE(58)NN2
  WRITE(6,1000)
  DC 100 I=1,NUMEL
  WRITE(6,1001)I
  DC 100 J=1,27
  K=KCONT(J,I)
100 WRITE(6,1002)J,K,XPT(K),YPT(K),ZPT(K)
1004 WRITE(6,1004)
  FCRMAT(//5X,'NODE COORDINATES BY NCDE NR'//5X,'NODE',5X,'
  1COORDINATES'//)
  CC 666 I=1,NN2
  
```


EAS023340
 EAS023350
 EAS023360
 EAS023370
 EAS023380
 EAS023390
 EAS023400
 EAS023410
 EAS023420
 EAS023430
 EAS023440
 EAS023450
 EAS023460
 EAS023470
 EAS023480
 EAS023490
 EAS023500
 EAS023510
 EAS023520
 EAS023530
 EAS023540
 EAS023550
 EAS023560
 EAS023570
 EAS023580
 EAS023590
 EAS023600
 EAS023610
 EAS023620
 EAS023630
 EAS023640
 EAS023650
 EAS023660
 EAS023670
 EAS023680
 EAS023690
 EAS023700
 EAS023710
 EAS023720
 EAS023730
 EAS023740
 EAS023750
 EAS023760
 EAS023770
 EAS023780
 EAS023790
 EAS023800

```

EMI( 1 ) = GP(R)*GP(S)*GP(T) - (EMI( 9) + EMI(12) + EMI(17))//TWC
EMI( 2 ) = GM(R)*GP(S)*GP(T) - (EMI( 9) + EMI(10) + EMI(18))//TWC
EMI( 3 ) = GM(R)*GM(S)*GP(T) - (EMI(10) + EMI(11) + EMI(19))//TWC
EMI( 4 ) = GM(R)*GM(S)*GP(T) - (EMI(11) + EMI(12) + EMI(20))//TWC
EMI( 5 ) = GP(R)*GP(S)*GM(T) - (EMI(12) + EMI(13) + EMI(17))//TWC
EMI( 6 ) = GM(R)*GP(S)*GM(T) - (EMI(13) + EMI(14) + EMI(18))//TWC
EMI( 7 ) = GM(R)*GM(S)*GM(T) - (EMI(14) + EMI(15) + EMI(19))//TWC
EMI( 8 ) = GP(R)*GM(S)*GM(T) - (EMI(15) + EMI(16) + EMI(20))//TWC
DC 700 I=1,3
PRCD1(I)=0, CDO
DC 700 J=1,20
PROCL(I)=PRCD1(I)+EMI(J)*COD(J,I)
XX=PROCL(1)
YY=PROCL(2)
ZZ=PROCL(3)
RETURN
END
  
```

700

SUBROUTINE ROADIN(XPT,YPT,ZPT,NCCNT,NREL,NUMPT)

SUBROUTINE TO READ ADINA DECK.

```

IMPLICIT REAL*8 (A-H,O-Z)
REAL*4 TITLE(20)
COMMON/ IPARA/ NUMNP, NUMEL
COMMON/ LOADS/ NLOAC, NLCUR, NPTM, ICGRAV
DIMENSION XPT(1), YPT(1), ZPT(1), NUMPT(1), NREL(1), NCCNT(20,1)
DIMENSION IDOF(6), ID(6), IDOLD(6), NCDE(20), NPAR(20)
1, NP(20), INF(20)
DATA CTEST/ *X
  
```

```

DATA CTEST/ *X
NCARD=0
READ(5,9000)(TITLE(I),I=1,20)
WRITE(6,9011)(TITLE(I),I=1,20)
FCRMTAT(///5X,20A4,///)
READ MASTER CONTROL CARDS
NUMNP = TOTAL NUMBER OF NODE POINTS
NELTYP = NUMBER OF ELEMENT GROUPS
READ(5,9001) NUMNP, IDOF(1), I=1,6, NEGL, NEGNL, MCODEX, NSTE
FCRMTAT(15,6I1,I4,3I5)
NELTYP=NEGL+NEGNL
NNODE=NUMNP
READ(5,7002) IMASS, ICAMP, IMASSN, ICAMPN
FCRMTAT(4I5)
READ(5,70C2) IEIG
READ(5,7002) ISREF, NUMREF, IEQUIT, ITEMAX
  
```

9011

C ***

C ***

9001

7002

C C C C C C C

EAS02820
 EAS02830
 EAS02840
 EAS02850
 EAS02860
 EAS02870
 EAS02880
 EAS02890
 EAS02900
 EAS02910
 EAS02920
 EAS02930
 EAS02940
 EAS02950
 EAS02960
 EAS02970
 EAS02980
 EAS02990
 EAS03000
 EAS03010
 EAS03020
 EAS03030
 EAS03040
 EAS03050
 EAS03060
 EAS03070
 EAS03080
 EAS03090
 EAS03100
 EAS03110
 EAS03120
 EAS03130
 EAS03140
 EAS03150
 EAS03160
 EAS03170
 EAS03180
 EAS03190
 EAS03200
 EAS03210
 EAS03220
 EAS03230
 EAS03240
 EAS03250
 EAS03260
 EAS03270
 EAS03280
 EAS03290

```

C ***
READ(5,5000) DUMMY
READ(5,5000) DUMMY
READ(5,5000) DUMMY
READ OR GENERATE NODAL POINT DATA
NCLD=0
NEG=0
10 READ(5,5006) CT,N,(ID(I),I=1,6),XPT(N),YPT(N),ZPT(N),KN
9006 FCRMAT(A1,I4,I1X,I4,5I5,3F10.0,I5)
C *** CHECK FOR CYLINDRICAL COORDINATES
IF(CT.NE.CTEST) GC TO 12
DUM=ZPT(N)/57.2958
R=YPT(N)
YPT(N)=R*DCOS(ZPT(N)/57.2958DD0)
ZPT(N)=R*DSIN(ZPT(N)/57.2958DD0)
12 CONTINUE
NLMPT(N)=N
IF(NOLD.EQ.C) GO TO 50
FCR GENERATION OF FIXED BOUNDARY CONDITIONS
CC 15 I=1,6
IF(IDOLD(I).EQ.-1.AND.ID(I).EQ.0) ID(I)=IDOLD(I)
CONTINUE
15 IF(KNOLD.EQ.0) GO TO 50
NLM=(N-NOLD)/KNOLD
NLMN=NUM-1
IF(NUMN.LT.1) GO TO 50
TC COUNT DOFS TO DETERMINE NUMBER CF IC CARDS
CC 20 I=1,6
IF(IDOF(I).EQ.0.AND.IDOLD(I).EQ.0) NEG=NEG+NUMN
CONTINUE
20 DX=(XPT(N)-XPT(NOLD))/NUM
IF(CT.NE.CTEST) GO TO 21
RCLD=YPT(NOLD)/DCOS(DUMOLD)
RNEW=YPT(N)/DCOS(DUM)
CR=(RNEW-ROLD)/NUM
DT=(LUM-DUMCLD)/NUM
GC TO 22
21 CCNTINUE
DY=(YPT(N)-YPT(NOLD))/NUM
DZ=(ZPT(N)-ZPT(NOLD))/NUM
CCNTINUE
22 K=NCLD
DC 30 J=1,NLMN
KK=K
K=K+KNOLC
XPT(K)=XPT(KK)+DX
IF(CT.NE.CTEST) GO TO 26
ROLD=ROLD+DR
DUMOLD=DUMOLD+DT
  
```



```

YPT(K)=ROLD*DCCS(DUMOLD)
ZPT(K)=ROLD*DSIN(DUMOLD)
GO TO 2E
CONTINUE
YPT(K)=YPT(KK)+DY
ZPT(K)=ZPT(KK)+DZ
CONTINUE
NUMPT(K)=K
CONTINUE
NCLD=N
KNCLD=KN
DUMOLD=DUM
TC COUNT DOFS TO DETERMINE NUMBER CF IC CARCS
DC 55 I=1,6
IF(IDOF(I).EQ.0.AND.ID(I).EQ.C) NEQ=NEQ+1
IF(IDOLD(I)=ID(I))
CONTINUE
IF(N.NE.NUMNP) GO TO 10
READ LOAD CCNTROL CARCS
FCR MAT(415)
DC 80 I=1,IMASSN
IF(IMASSN.EQ.0) GO TO 81
READ(5,9000) DUMMY
CONTINUE
CONTINUE
IF(IDAMPN.EQ.0) GO TO 91
DC 90 I=1,ICAMPN
READ(5,9000) DUMMY
CONTINUE
CONTINUE
READ INITIAL CONDITIONS
READ(5,9002) ICON
IF(ICON.EQ.C) GO TO 100
CARDNR=NEQ/6.0
NCARD=IDINT(CARDNR)
TEST=CARDNR-NCARD
IF(TEST.GT.0.1) NCARD=NCARD+1
DC 95 I=1,NCARD
READ(5,9000) DUMMY
CONTINUE
IF(IMASS.EQ.0) GO TO 103
DC 96 I=1,NCARD
READ(5,9000) DUMMY
CONTINUE
DC 98 I=1,NCARD
READ(5,9000) DUMMY
CONTINUE

```



```

9007 FCRMAT(6E12.6)
100 CCNTINUE
    WRITE(6,9009) NEQ,NCARD
    FCRMAT(///,NEQ,NCARD FOR IC IN GEOM1 = ',I5,10X,I5///)
C *** READ ELEMENT CARDS
DO 900 M=1,NELTYP
    READ(5,9008) (NPAR(I),I=1,20)
    FCRMAT(20I4) (NPAR(I),I=1,20)
9008 FCRMAT(20I4)
9010 FCRMAT(///,NPAR = ',20I5///)
    MTYPE=NPAR(1)
    NUMMAT=NPAR(16)
    NSTRES=NPAR(13)
C *** CALCULATE THE NUMBER OF MATERIAL CASE CARDS
    IF(NPAR(15).EQ.1) NCARD=1
    IF(NPAR(15).EQ.2) NCARD=2+NPAR(18)
    IF(NPAR(15).EQ.3) NCARD=4
    IF(NPAR(15).EQ.4) NCARD=4
    IF(NPAR(15).EQ.5) NCARD=2
    IF(NPAR(15).EQ.8) NCARD=1
    IF(NPAR(15).EQ.9) NCARD=1
    IF(NPAR(15).EQ.10) NCARD=6
    IF(NPAR(15).EQ.11) NCARD=6
    IF(NPAR(15).EQ.12) GC TO 111
    CARDNR=NPAR(17)/8.0
    NCARD=ICINT(CARDNR)
    TEST=CARDNR-NCARD
    IF(TEST.GT.C 1) NCARD=NCARD+1
111 CCNTINUE
C *** READ MATERIAL PROPERTIES
DO 222 J=1,NUMMAT
    READ(5,9000) DUMMY
9000 FCRMAT(20A4)
    DO 45 I=1,NCARD
        READ(5,9000) DUMMY
        CCNTINUE
45 CCNTINUE
C *** READ STRESS OUTPUT TABLE CARDS
    IF(NPAR(13).EQ.0) GO TO 61
    DO 60 I=1,NSTRES
        READ(5,9000) DUMMY
        CCNTINUE
60 CCNTINUE
61 IF(NPAR(14).EQ.0) NPAR(14)=1
    NEL=NPAR(14)-1
120 READ(5,9002) INEL,IINC

```


EAS04260
 EAS04270
 EAS04280
 EAS04290
 EAS04300
 EAS04310
 EAS04320
 EAS04330
 EAS04340
 EAS04350
 EAS04360
 EAS04370
 EAS04380
 EAS04390
 EAS04400
 EAS04410
 EAS04420
 EAS04430
 EAS04440
 EAS04450
 EAS04460
 EAS04470
 EAS04480
 EAS04490
 EAS04500
 EAS04510
 EAS04520
 EAS04530
 EAS04540
 EAS04550
 EAS04560

```

9002 FCRMAT(I5,3CX,I5)
      IF(IINC.EQ.C) IINC=1
      READ(5,9004) (INP(I),I=1,8)
      READ(5,9004) (INP(I),I=9,N20)

9004 FCRMAT(12,I5)
140  NEL=NEL+1
      NREL(NEL)=NEL
      ML=INEL
      IF(ML) 150,155,16C
150  WRITE(6,800C)NEL
800C FCRMAT(/,5X,'WARNING - ERROR IN ELEMENT ',I5//)
C *** NC GENERATION OF NCDE POINTS REQUIRED
155  DC 156 I=1,N20
      NP(I)=INP(I)
      NCONT(I,NEL)=NP(I)
      CONTINUE

156  GC TO 162
C ** GENERATION OF NODE POINTS REQUIRED
160  DC 161 I=1,N20
      IF(NP(I).EQ.0) GO TO 161
      NP(I)=NP(I)+KA
      NCONT(I,NEL)=NP(I)
      CONTINUE

161  CCNTINUE
162  NUMEL=NUMEL+1
      IF(NEL.EQ.NPAR(2)) RETURN
      IF(NEL.LT.INEL) GO TO 140
      KA=IINC
      GC TO 130
900  CONTINUE
      END
  
```


APPENDIX E

PROGRAM STRESS LISTING

```

** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** ** **  PROGRAM STRESS
** ** ** ** **  THIS PROGRAM TAKES THE CONNECTIVITY DATA FROM PROGRAM
** ** ** **  KCONT AND THE STRESS RESULTS FROM AN ADINA ANALYSIS WITH
** ** ** **  THE 27 NODE BRICK CUPOT OPTION USED AND AVERAGES THE
** ** ** **  CONTRIBUTIONS TO EACH NODE BY THE VARIOUS ELEMENTS TO
** ** ** **  WHICH IT IS A MEMBER. THESE VALUES ARE THEN OPERATED ON
** ** ** **  TO YIELD THE THREE PRINCIPAL STRESSES AND AN ARRAY OF THE
** ** ** **  CONTAINING THE MAXIMUM PRINCIPAL STRESS FOR EACH NODE. THE
** ** ** **  CUPOT CCNSISTS OF A LINE PRINTER LISTING OF AVERAGE AND
** ** ** **  PRINCIPAL STRESSES AND STORAGE CN USER DEFINED FILES 60
** ** ** **  AND 61 RESPECTIVELY.
** ** ** **  OUTPUT STORAGE:
** ** ** **  FT60FC01-SEQUENTIAL LISTING OF THE AVERAGE STRESS ARRAYS
** ** ** **  IN THE FOLLOWING ORDER: ASIGX,ASIGY,ASIGZ,ATAUXY,
** ** ** **  ATAUXZ,ATAUYZ.
** ** ** **  FT61FC01-SEQUENTIAL LISTING OF THE PRINCIPAL STRESS ARRAYS
** ** ** **  IN THE FOLLOWING ORDER: PSIG1,PSIG2,PSIG3,PMAX.
** ** ** **  INPUT DATA REQUIREMENTS:
** ** ** **  CARD 1      C COLUMN      REMARKS
** ** ** **  VARIABLE      1-5      NUMBER OF NODES IN 27 NODE BRICK MESH
** ** ** **  NN2          6-10      NUMBER OF ELEMENTS IN MESH
** ** ** **  NUMEL
** ** ** **  DD CARDS ARE REQUIRED FOR FILE 57 ESTABLISHED BY ADINA
** ** ** **  AND FILES 58 AND 59 ESTABLISHED BY PROGRAM KCONT.
** ** ** **  WRITTEN BY: L.R. EASTERLING NPS, MONTEREY CA., FEB 1978
** ** ** **  DIMENSION A(68000)
** ** ** **  DATA A/68000*0./
** ** ** **  MTCT=68000
** ** ** **  READ(5,1000)NN2,NUMEL
** ** ** **  N1=1

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STR000010
STR000020
STR000030
STR000040
STR000050
STR000060
STR000070
STR000080
STR000090
STR000100
STR000110
STR000120
STR000130
STR000140
STR000150
STR000160
STR000170
STR000180
STR000190
STR000200
STR000210
STR000220
STR000230
STR000240
STR000250
STR000260
STR000270
STR000280
STR000290
STR000300
STR000310
STR000320
STR000330
STR000340
STR000350
STR000360
STR000370
STR000380
STR000390
STR000400

CC


```

2=N1+27*NUMEL*2
N3=N2+27*NUMEL*2
N4=N3+27*NUMEL*2
N5=N4+27*NUMEL*2
N6=N5+27*NUMEL*2
N7=N6+27*NUMEL*2
N8=N7+NN2*2
N9=N8+NN2*2
N10=N9+NN2*2
N11=N10+NN2*2
N12=N11+NN2*2
N13=N12+NN2*2
N14=N13+NN2*2
N15=N14+NN2*2
N16=N15+NN2*2
N17=N16+NN2*2
N18=N17+3*NN2*2
N19=N18+NN2
NLAST=N19+27*NUMEL
IF(NLAST-GE.MTOT)GO TO 10
CALL STRESS(A(N1),A(N2),A(N3),A(N4),A(N5),A(N6),A(N7),A(N8),
1A(N9),A(N10),A(N11),A(N12),A(N13),A(N14),A(N15),A(N16),A(N17),
2A(N18),A(N19),NN2,NUMEL)
STOP
10 WRITE(6,1001)NLAST
1000 FCRMAT(215)
1001 FORMAT(///5X,'*****MTCT INSUFFICIENT-MUST BE GREATER THAN',
111001)
STOP
END
SUBROUTINE STRESS (SIGX,SIGY,SIGZ,TAUXY,TAUXZ,TAUXZ,TAUXZ,ASIGX,ASIGY,AS
1IGZ,ATAUXY,ATAUXZ,ATAUXZ,PSIG1,PSIG2,PSIG3,FMAX,COD,KTR,KCCNT,
2NN2,NUMEL)
IMPLICIT REAL*8(A-H,O-Z)
DIMENSION SIGX(27,NUMEL),TAUXZ(27,NUMEL),ASIGX(NN2),
1TAUXY(27,NUMEL),TAUXZ(27,NUMEL),ATAUXZ(NN2),ATAUXZ(NN2),
2ASIG1(NN2),ASIG2(NN2),PSIG3(NN2),FMAX(NN2),CCC(3,NN2),
3PSIG1(NN2),PSIG2(NN2),PSIG3(NN2),KTR(1),KCON(27,1)
DIMENSION KTR(1),KCON(27,1)
REWIND 59
REWIND 58
REWIND 57
READ(59)((KCON(I,J),J=1,NUMEL),I=1,27)
READ(58)NN2
CC 10 I=1,NUMEL
READ(57)IEL
CC 10 J=1,27
READ(57)NDUM,SIGX(J,IEL),SIGY(J,IEL),SIGZ(J,IEL),TAUXY(J,IEL),

```

STR00410
STR00420
STR00430
STR00440
STR00450
STR00460
STR00470
STR00480
STR00490
STR00500
STR00510
STR00520
STR00530
STR00540
STR00550
STR00560
STR00570
STR00580
STR00590
STR00600
STR00610
STR00620
STR00630
STR00640
STR00650
STR00660
STR00670
STR00680
STR00690
STR00700
STR00710
STR00720
STR00730
STR00740

STR00790
STR00800
STR00810
STR00820
STR00830
STR00840
STR00850
STR00860
STR00870
STR00880


```

1  TAUZX(J,IEL),TAUYZ(J,IEL)
10  CCNTINUE
20  CC 20 I=1,NN2
    READ(58)J,(CCD(K,J),K=1,3)
    DC 30 I=1,NN2
    ASIGX(I)=0.000
    ASIGY(I)=0.000
    ASIGZ(I)=0.000
    ATAUZY(I)=0.000
    ATAUZX(I)=0.000
    ATAUZZ(I)=0.000
    KTR(I)=0
30  DC 40 I=1,NLMEL
    DO 40 J=1,27
    K=KCONT(J,I)
    ASIGX(K)=ASIGX(K)+SIGX(J,I)
    ASIGY(K)=ASIGY(K)+SIGY(J,I)
    ASIGZ(K)=ASIGZ(K)+SIGZ(J,I)
    ATAUZY(K)=ATAUZY(K)+TAUZY(J,I)
    ATAUZX(K)=ATAUZX(K)+TAUXZ(J,I)
    ATAUZZ(K)=ATAUZZ(K)+TAUZZ(J,I)
    KTR(K)=KTR(K)+1
40  CC 50 I=1,NN2
    ASIGX(I)=ASIGX(I)/KTR(I)
    ASIGY(I)=ASIGY(I)/KTR(I)
    ASIGZ(I)=ASIGZ(I)/KTR(I)
    ATAUZY(I)=ATAUZY(I)/KTR(I)
    ATAUZX(I)=ATAUZX(I)/KTR(I)
    ATAUZZ(I)=ATAUZZ(I)/KTR(I)
50  REWIND 60
    WRITE(60)ASIGX,ASIGY,ASIGZ,ATAUZY,ATAUZX,ATAUZZ
    WRITE(6,1001)
    FORMAT(///5X,'AVERAGED NODAL STRESSES'////)
1001  CC 60 I=1,NN2
    WRITE(6,1000)I,ASIGX(I),ASIGY(I),ASIGZ(I),ATAUZY(I),
    ATAUZX(I),ATAUZZ(I)
1  CCNTINUE
60  FCRMAT(1,X,I6,6F16.5)
1000  CTHRD=1.000/3.000
    PI=3.141592654
    DC 100 I=1,NN2
    P=-((ASIGX(I)+ASIGY(I)+ASIGZ(I))
    Q=ASIGX(I)*ASIGY(I)+ASIGY(I)*ASIGZ(I)+ASIGZ(I)*ASIGX(I)
    1  R=-ATAUZY(I)*2-ATAUZX(I)*2-ATAUZZ(I)*2
    1  R=-(ASIGX(I)*ASIGY(I)+ASIGY(I)*ASIGZ(I)+ASIGZ(I)*ASIGX(I)
    1  R=-(ATAUZY(I)*ATAUZX(I)+ATAUZX(I)*ATAUZZ(I)+ATAUZZ(I)*
    22)
    A=(3.000*(C-F**2)/3.000

```

STR000890
 STR000900
 STR000910
 STR000920
 STR000930
 STR000940
 STR000950
 STR000960
 STR000970
 STR000980
 STR000990
 STR010000
 STR010010
 STR010020
 STR010030
 STR010040
 STR010050
 STR010060
 STR010070
 STR010080
 STR010090
 STR010100
 STR010110
 STR010120
 STR010130
 STR010140
 STR010150
 STR010160
 STR010170
 STR010180
 STR010190
 STR010200
 STR010210
 STR010220
 STR010230
 STR010240
 STR010250
 STR010260
 STR010270
 STR010280
 STR010290
 STR010300
 STR010310
 STR010320
 STR010330
 STR010340
 STR010350
 STR010360


```

200  B=(2.0D0*P**3-9.0D0*P*Q+27.0D0*R)/27.0D0
      AE=(B**2)/4.0D0+(A**3)/27.0D0
      IF (AB)400,300,200
      WRITE(6,2000)I,AB,A,B,P,Q,R
      PSIG1(I)=1111111.
      PSIG2(I)=PSIG1(I)
      PSIG3(I)=PSIG1(I)
      GO TO 100
300  AA=(-B/2.0D0)**0.5THRD
      BE=AA
      PSIG1(I)=AA*2.0D0-P/3.0D0
      PMAX(I)=PSIG1(I)
      PSIG2(I)=-AA-P/3.0D0
      PSIG3(I)=PSIG2(I)
      IF (PSIG2(I).GT.PSIG1(I))PMAX(I)=PSIG2(I)
      GO TO 100
400  PHI=DARCO5((-B/2.0D0)/DSQRT((-A**3)/27.0D0))
      PSIG1(I)=2.0D0*DSQRT(-A/3.0D0)*DCCS(PHI/3.0D0)-P/3.0D0
      PSIG2(I)=2.0D0*DSQRT(-A/3.0D0)*DCCS(PHI/3.0D0*PI/3.0D0)
      1 -P/3.0D0
      1 PSIG3(I)=2.0D0*DSQRT(-A/3.0D0)*DCCS(PHI/3.0D0*PI/3.0D0)
      1 -P/3.0D0
      PMAX(I)=PSIG1(I)
      IF (PSIG2(I).GT.PMAX(I))PMAX(I)=PSIG2(I)
      IF (PSIG3(I).GT.PMAX(I))PMAX(I)=PSIG3(I)
      CGCONTINUE
100  REWIND 61
      WRITE(61)PSIG1,PSIG2,PSIG3,PMAX
      WRITE(6,2001)
      CC 600 I=1,KN2
600  FCRMAT(6,2002)I,PMAX(I),PSIG1(I),PSIG2(I),PSIG3(I),
2000  1 10X,AB=,G25.16/10X,A=,G25.16/10X,B=,G25.16/
      2 10X,P=,G25.16/10X,Q=,G25.16/10X,R=,G25.16///)
2001  FCRMAT(//20X,NODAL PRINCIPAL STRESS FOR FIRST STAGE ROTCR ELADE
2002  1 10X,NODE,5X,MAXIMUM,13X,NR 1,16X,NR 2,16X,NR 3,/)
      FCRMAT(1X,1E,5X,4G20.11)
      ENDFILE 61
      RETURN
      END

```

STR011370
 STR011380
 STR011390
 STR011400
 STR011410
 STR011420
 STR011430
 STR011440
 STR011450
 STR011460
 STR011470
 STR011480
 STR011490
 STR011500
 STR011510
 STR011520
 STR011530
 STR011540
 STR011550
 STR011560
 STR011570
 STR011580
 STR011590
 STR011600
 STR011610
 STR011620
 STR011630
 STR011640
 STR011650
 STR011660
 STR011670
 STR011680
 STR011690
 STR011700
 STR011710
 STR011720
 STR011730
 STR011740
 STR011750
 STR011760
 STR011770

APPENDIX F

PROGRAM CONTOUR PLOT DATA LISTING

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*****
PROGRAM CCNTOUR PLOT DATA
THIS PROGRAM IS DESIGNED TO TAKE NCCLAL PCINT DATA FROM
AN ADINA 3-D MESH FROM FILE 58 OF PRGCRAM KCCNT AND PLOT
NODE LOCATION MARKS AND NODE NUMBERS FOR VARIOUS DESIRED
2-DIMENSIONAL PLANES. PLANES MAY BE DEFINED BY SPECIFIC
COORDINATE VALUE, LIMITING COORDINATE VALUES, OR INPUT
OF DESIRED NODE NUMBERS TO BE PLOTTED. OUTPUT
DATA FOR CONTOUR PLOTTING CAN ALSO BE OUTPUT
ON THE LINE PRINTER AND/OR PUNCHED CARDS.

REQUIRED DATA DECK:
CONTROL CARD 1 (FORMAT 4110)

VARIABLE      COL
NN2            1-10
NPIC           11-20
NDMAX          21-30
NPCH           31-40

REMARKS
TOTAL NUMBER OF NODES
TOTAL NUMBER OF
DIFFERENT PLANES TO BE
PLOTTED
HIGHEST NUMBER OF
NODES EXPECTED PER
PLANE
FLAG FOR PUNCHING
CONTOUR PLOT DATA
0 FOR PUNCHING
1 FOR NO PUNCH

NODE CARDS
INPUT NPIC SETS OF CARDS. LEAVE COLUMNS FOR
TESTL, TESTH, ICRD BLANK IF NODES TO BE READ IN.
FOR EQUALITY TEST INPUT TESTL=TESTH.

CARD1 (FORMAT 20A4) TITLE CARD FOR ANALYSIS
PLANE.
*****

```

CC

[illegible]

CARD 2 (FORMAT 2F10,3I5)

TESTF	1-10
TESTH	11-20
ICRD	21-25

IRD 26-30

ICT	31-35
MX	11-11
MY	16-20

IF IRD=1, PLACE HERE SUFFICIENT
NODE NUMBERS TO BE PLOTTED USING
CARD 3-PLCT TITLE CARD FORMAT 20A4 A
TO BE PLOTTED BELOW FIRST PLOT A
PLOT. MAXIMUM OF 80 CHARACTERS.

CARD 4 (FORMAT F10.4)

DMAGS 1-10

MAGNIFICATION OF
COORDINATE VALUE SO
THAT MAXIMUM DIMEN-
SION DOES NOT
EXCEED 9 IN.
1=X CGCRD
2=Y CGCRD
3=Z CGCRD

WRITTEN BY: LAEL R. EASTERLING
NPS, MONTEREY CA, FEBRUARY 1978

[illegible]

MAIN PROGRAM TO DETERMINE
DIMENSIONS AND CALL SUB-
ROUTINES

DIMENSION A(26000)
COMMON/ IPARA/ NPCH
MTCT=26000
READ(5,1000) NN2, NPIC, NDMAX, NPCH

N1=1+NN2*2
N2=2+NN2*2
N3=3+NN2*2
N4=4+NN2*2
N5=5+NN2*2
N6=6+NN2*2
N7=7+NN2*2
N8=8+NN2*2
N9=9+NDMAX*NPIC
N10=N9+3*NN2
N11=N10+NPIC
N12=N11+3*NN2
N13=N12+NPIC*20
N14=N13+2*NPIC
IF(NLAST.GT.MTCT) GO TO 111
CALL DEFPLT (A(N1),A(N2),A(N3),A(N4),A(N5),A(N6),A(N7),
1A(N8),A(N9),A(N10),A(N11),A(N12),A(N13),NN2,NPIC,NDMAX)
STOP
111 WRITE(6,2000) NLAST
2000 FORMAT(//1X,'NLAST= ',110,'MTCT INSUFFICIENT'//)
1000 FCFORMAT(4110)
STOP
END

CCCCCCCC

SUBROUTINE TO DEVELOP ANA-
LYSIS PLANES AND CALL PLOT
ROUTINE

SUBROUTINE DEFPLT (XX,YY,ZZ,SIG1,SIG2,SIG3,SIGMX,LVL,COORD,KTR,
1PCOORD,TITLE,MC,NN2,NPIC,NDMAX)
REAL*8 XX(1),YY(1),ZZ(1),SIG1(1),SIG2(1),SIG3(1),SIGMX(1)
DIMENSION LVL(NPIC,1),COORD(NN2,1),KTR(1),FCCORD(NN2,1)
DIMENSION TITLE(NPIC,1),MC(2,1)

CTRP0890
CTRP0900
CTRP0910
CTRP0920
CTRP0930
CTRP0940
CTRP0950
CTRP0960
CTRP0970
CTRP0980
CTRP0990
CTRP1000
CTRP1010
CTRP1020
CTRP1030
CTRP1040
CTRP1050
CTRP1060
CTRP1070
CTRP1080
CTRP1090
CTRP1100
CTRP1110
CTRP1120
CTRP1130
CTRP1140
CTRP1150
CTRP1160
CTRP1170
CTRP1180
CTRP1190
CTRP1200
CTRP1210
CTRP1220
CTRP1230
CTRP1240
CTRP1250
CTRP1260
CTRP1270
CTRP1280
CTRP1290
CTRP1300
CTRP1310
CTRP1320
CTRP1330
CTRP1340
CTRP1350
CTRP1360

CTRP1370
 CTRP1380
 CTRP1390
 CTRP1400
 CTRP1410
 CTRP1420
 CTRP1430
 CTRP1440
 CTRP1450
 CTRP1460
 CTRP1470
 CTRP1480
 CTRP1490
 CTRP1500
 CTRP1510
 CTRP1520
 CTRP1530
 CTRP1540
 CTRP1550
 CTRP1560
 CTRP1570
 CTRP1580
 CTRP1590
 CTRP1600
 CTRP1610
 CTRP1620
 CTRP1630
 CTRP1640
 CTRP1650
 CTRP1660
 CTRP1670
 CTRP1680
 CTRP1690
 CTRP1700
 CTRP1710
 CTRP1720
 CTRP1730
 CTRP1740
 CTRP1750
 CTRP1760
 CTRP1770
 CTRP1780
 CTRP1790
 CTRP1800
 CTRP1810
 CTRP1820
 CTRP1830
 CTRP1840

```

CMMON/IPARA/NPCH
DC 10 I=1,NPCT
KTR(I)=0
DC 10 J=1,NDMAX
LVL(I,J)=0
READ(58)DUM
DC 20 I=1,NN2
READ(58)IDUM,XX(I),YY(I),ZZ(I)
DC 35 I=1,NN2
CCORD(I,1)=XX(I)
CCORD(I,2)=YY(I)
CCORD(I,3)=ZZ(I)
CCNTINUE
DC 30 I=1,NPCT
READ(5,1002)(TITLE(I,JJ),JJ=1,20)
FCRMT(20A4)
1002 READ(5,1000)TESTL,TESTH,ICRD,IRD,ICT,MC(1,I),MC(2,I)
1000 FCRMT(2F10.0,5I5)
IF(IRD.EQ.1)GO TO 111
DC 40 J=1,NN2
IF(CCORD(J,ICRD).GT.TESTL.AND.CCORD(J,ICRD).LT.TESTH.OR.
1CCORD(J,ICRD).EQ.TESTL)GO TO 222
GC TO 40
222 KTR(I)=KTR(I)+1
K=KTR(I)
LVL(I,K)=J
CCNTINUE
GC TO 30
111 READ(5,1001)(LVL(I,L),L=1,ICT)
1001 FCRMT(16I5)
KTR(I)=ICT
30 CCNTINUE
WRITE(6,200C)
DC 45 I=1,NPCT
K=KTR(I)
45 WRITE(6,2001)I,(LVL(I,J),J=1,K)
CALL MYPLOT(LVL,COORD,KTR,PCCOORD,MC,NN2,NPCT,NDMAX)
CALL PLTDT(SIG1,SIG2,SIG3,SIGMX,LVL,COORD,KTR,TITLE,MC,
1NN2,NPCT)
2000 FCRMT(//5X,'NODES NUMBERS BY ANALYSIS PLANE'///)
2001 FCRMT(//5X,'PLANE NR ',I5//1X,4(1X,25I5//)
END
  
```

C
 C
 C

C

SUBROUTINE TG PLCT ANALYSIS PLANES

```

SUBROUTINE MYPLOT(LVL,COORD,KTR,PCCORD,MC,NN2,NP1CT,NDMAX)
DIMENSION LVL(NP1CT,1),KTR(1),PCCORD(NN2,1),FCCCRD(NN2,1)
DIMENSION TITLE1(20)
DIMENSION MC(2,1)
COMMON/ IPARA/NPCH
READ(5,1000)(TITLE1(I),I=1,20)
READ(5,1001)DMAGS
CC 10 I=1,NN2
CC 10 J=1,3
10 PCCORD(I,J)=COORD(I,J)*DMAGS
CALL PLOTS
CALL PLOT(-12.,3.,-3)
CALL PLOT(1.,0.,-3)
CALL SYMBOL(0.,0.,14,TITLE1,0.,8C)
CALL PLOT(4.25,5.25,-3)
CC 20 I=1,NP1CT
MX=MC(1,1)
MY=MC(2,1)
K=KTR(1)
CC 30 J=1,K
L=LVL(I,J)
X=PCOORD(L,MX)-.02
Y=PCOORD(L,MY)-.035
CALL SYMBOL(X,Y,.C7,'X',0.,1)
X=X+.06
Y=Y-.035
F=FLOAT(L)
CALL NUMBER(X,Y,.14,F,0.,-1)
CCNT INUE
30 CALL PLCT(-4.25,5.,-3)
CALL SYMBOL(0.,0.,14,LEVEL,0.,5)
F=FLOAT(I)
CALL NUMBER(1.0,0.,.14,F,0.,-1)
CALL PLCT(4.25,5.25,-3)
20 CCNT INUE
CALL SYMBOL(-4.25,0.,.14,TITLE1,0.,15)
CALL PLCT(-4.25,5.,-3)
CALL PLOT
CALL PLOT
1000 FCRMAT(20A4)
1001 RETURN
END

```

C

CTRP1850
 CTRP1860
 CTRP1870
 CTRP1880
 CTRP1890
 CTRP1900
 CTRP1910
 CTRP1920
 CTRP1930
 CTRP1940
 CTRP1950
 CTRP1960
 CTRP1970
 CTRP1980
 CTRP1990
 CTRP2000
 CTRP2010
 CTRP2020
 CTRP2030
 CTRP2040
 CTRP2050
 CTRP2060
 CTRP2070
 CTRP2080
 CTRP2090
 CTRP2100
 CTRP2110
 CTRP2120
 CTRP2130
 CTRP2140
 CTRP2150
 CTRP2160
 CTRP2170
 CTRP2180
 CTRP2190
 CTRP2200
 CTRP2210
 CTRP2220
 CTRP2230
 CTRP2240
 CTRP2250
 CTRP2260
 CTRP2270
 CTRP2280
 CTRP2290
 CTRP2300
 CTRP2310
 CTRP2320

CTRP2330
 CTRP2340
 CTRP2350
 CTRP2360
 CTRP2370
 CTRP2380
 CTRP2390
 CTRP2400
 CTRP2410
 CTRP2420
 CTRP2430
 CTRP2440
 CTRP2450
 CTRP2460
 CTRP2470
 CTRP2480
 CTRP2490
 CTRP2500
 CTRP2510
 CTRP2520
 CTRP2530
 CTRP2540
 CTRP2550
 CTRP2560
 CTRP2570
 CTRP2580
 CTRP2590
 CTRP2600
 CTRP2610
 CTRP2620
 CTRP2630
 CTRP2640
 CTRP2650
 CTRP2660
 CTRP2670
 CTRP2680
 CTRP2690
 CTRP2700
 CTRP2710
 CTRP2720
 CTRP2730
 CTRP2740
 CTRP2750
 CTRP2760
 CTRP2770
 CTRP2780
 CTRP2790
 CTRP2800

SUBROUTINE TO PRINT AND PUNCH CONTOUR
PLOT DATA

```

SUBROUTINE PLTDT(SIG1,SIG2,SIG3,SIGMX,LVL,COORD,KTR,TITLE,MC,
1  IAN2,NPICT)
  REAL*8 SIG1(SIG2),SIG2(NN2),SIG3(NN2),SIGMX(NN2)
  DIMENSION LVL(NPICT,1),COORD(NN2,1),KTR(1)
  COMMON/IPARA/NPCH
  REWIND 61
  READ(61)SIG1,SIG2,SIG3,SIGMX
  DO 10 I=1,NPICH
    MX=MC(1,1)
    MY=MC(2,1)
    IF(NPCH.EQ.0)WRITE(7,3002)(TITLE(I,JJ),JJ=1,20)
    FGMAT(20A4)
    3002 K=KTR(I)
    IF(NPCH.EQ.0)WRITE(7,3000)
    WRITE(6,200C)I
    3003 WRITE(6,3003)(TITLE(I,JJ),JJ=1,20)
    FCRMAT(/5X,20A4//)
    CC 20 J=1,K
    L=LVL(I,J)
    WRITE(6,2001)L,COORD(L,MX),COORD(L,MY)
    IF(NPCH.EQ.0)WRITE(7,2002)L,COORD(L,MX),COORD(L,MY)
    20 CCNTINUE
    IF(NPCH.EQ.C)WRITE(7,3001)
    WRITE(6,2003)
    DC 30 J=1,K
    L=LVL(I,J)
    WRITE(6,2004)L,SIGMX(L)
    IF(NPCH.EQ.0)WRITE(7,2005)L,SIGMX(L)
    30 CCNTINUE
    10 FGMAT(/5X,'COORDINATES OF NODES TO BE USED IN CONTOUR
    PLOT--PLANE ',I5//)
    2001 FCRMAT(1X,I5,2F15.5)
    2002 FCRMAT(15,2F15.5)
    2003 FCRMAT(/5X,'STRESSES TO BE USED IN CONTOUR PLCT'//)
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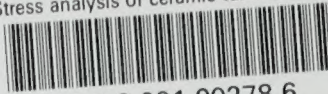
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